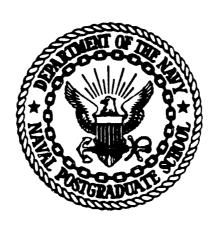
HATTE FUSIONALUATE SCHOOL MONTEREY CA F/G 20/4 EFFECTS OF OSCILLATION FREQUENCY AND AMPLITUDE ON SEPARATION IN--ETC(U) SEP 80 M FOX AD-A096 386 UNCLASSIFIED NL 1 or 2 AD A 388



NAVAL POSTGRADUATE SCHOOL

Monterey, California





THESIS,

EFFECTS OF OSCILLATION FREQUENCY
AND AMPLITUDE ON SEPARATION IN
AN UNSTEADY TURBULENT FLOW

by

16 Martin Fox

12/10

// September 1980

Thesis Advisor:

J. A. Miller

Approved for public release; distribution unlimited

Unclassified

SECURITY CLASSIFICATION OF THIS PAGE (When Date Entered)

REPORT DOCUMENTATION PAGE	READ INSTRUCTIONS BEFORE COMPLETING FORM						
APPOAT NUMBER 2. GOVT ACCESSION NO AD-A096 386							
Effects of Oscillation Frequency and Amplitude on Separation in	Master's Thesis; September, 1980						
an Unsteady Turbulent Flow	6. PERFORMING ORG. REPORT NUMBER						
Martin Fox	S. CONTRACT OR GRANT NUMBER(s)						
PERFORMING ORGANIZATION NAME AND ADDRESS	10. PROGRAM ELEMENT PROJECT TASK						
Naval Postgraduate School Monterey, California 93940	15. PROGRAM ELEMENT, PROJECT, TASH AREA & WORK UNIT NUMBERS						
CONTROLLING OFFICE NAME AND ADDRESS	12. REPORT DATE September, 1980						
Naval Postgraduate School Monterey, California 93940	13. NUMBER OF PAGES						
4 MONITORING AGENCY NAME & ADDRESS(II different from Controlling Office)	18. SECURITY CLASS. (of this report)						
Naval Postgraduate School Monterey, California 93940	Unclassified						
////	The. DECLASSIFICATION/DOWNGRADING						

Approved for public release; distribution unlimited

17. DISTRIBUTION STATEMENT (of the sharest entered in Block 29, Il different from Report)

16. SUPPLEMENTARY NOTES

19. KEY WORDS (Continue on reverse side if necessary and identity by block number)

Turbulent Boundary Layer Oscillating Flow

20. ABSTRACT (Continue on reverse side if necessary and identify by black member)

A two-dimensional model was developed and used in a preliminary investigation of the relationship between flow oscillation frequency, oscillation amplitude, and turbulent boundary layer separation in a low speed, oscillating wind tunnel. It was found that the frequency of oscillation had a

profound effect upon the amplitude of oscillation and flow separation. Frequencies from 20 Hz to 28 Hz and 70 Hz to 80 Hz

DD | JAN 73 1473 (Page 1)

EDITION OF 1 MOV 65 IS OBSOLETE 5/N 0102-014-4401

Unclassified
SECURITY CLASSIFICATION OF THIS PAGE (Then Date Entered)

20. ABSTRACT (continued)

allowed attachment of the boundary layer, while other frequencies, up to 100 Hz, caused flow separation in an eighteen degree divergent section.

Accession F	or
TTIS GRAND	
1 ·	
ANG	_es
Dist 100	31
1	

DD Form 1473 S/N 0102-014-6601 Approved for public release; distribution unlimited

Effects of Oscillation Frequency and Amplitude on Separation in an Unsteady Turbulent Flow

bу

Martin Fox Lieutenant, United States Navy B.S.A.E., Auburn University, 1973

Submitted in partial fulfillment of the requirements for the degree of

MASTER OF SCIENCE IN AERONAUTICAL ENGINEERING

from the

NAVAL POSTGRADUATE SCHOOL September, 1980

Approved by:

My F. Plate

Chairman, Department of Aeronautics

Melliam M. Jelles

Dean of Science and Engineering

ABSTRACT

A two-dimensional model was developed and used in a preliminary investigation of the relationship between flow oscillation frequency, oscillation amplitude, and turbulent boundary layer separation in a low speed, oscillating wind tunnel.

It was found that the frequency of oscillation had a profound effect upon the amplitude of oscillation and flow separation. Frequencies from 20 Hz to 28 Hz and 70 Hz to 80 Hz allowed attachment of the boundary layer, while other frequencies, up to 100 Hz, caused flow separation in an eighteen degree divergent section.

TABLE OF CONTENTS

LIST	OF T	ABLES	•	•	•	•	•	•	•	•	•	•	•	•	•	•	•	•	•	•	•	•	7
LIST	OF F	'IGURES		•	•	•	•	•	•	•	•	•		•	•	•	•	•	•	•	•	•	8
LIST	OF S	YMBOLS				•	•	•		•	•	•	•	•	•	•	•	•	•	•	•		10
ACKNO	WLED	GEMENT	rs	•		•	•		•	•	•	• (•	•	•	•	•	•	•	•	•		11
I.	Int	RODUCI	IOI	N	•	•	•	•		•	•	•		•	•		•	•	•	•	•	•	12
II.	EXP	ERIMEN	AT!	LE	EQI	JIF	ME	NT	1	•		•		•	•	•		•	•	•	•	•	14
	Α.	oscII	LLA!	rin	IG	FI	COM	r W	IN	ID	TU	INN	ŒĬ	٠	•			•	•	•	•		14
		1. 0	en	era	al	٦e	sc	ri	.pt	ii	on	•	•	•	•	•		•	•	•	•	•	14
		2. I	10	N (sc	il	.la	ti	.or	1 5	Sys	ite	em	•	•	•			•		•	•	15
	В.	MODEI	S	•	•	•	•		•	•	•	•	•	•	•	•	•	•	•	•	•	•	20
		1. I	Pre	lir	nir	nar	У	Mo	de	218	3	•	•	•	•	•	•	•	•	•	•	•	20
		2. I	ri	naı	ту	Mo	ode	els	;	•	•	•	•	•	•	•	•	•	•	•	•	•	20
	c.	Inst	RUM	ENI	ra:	PIC	N	•	•	•	•	•	•	•	•	•	•	•	•	•	•	•	24
		1. I	re	est	tre	an	n F	lo	w	Se	ens	302	. .	•	•	•	•	•	•	•	•	•	24
		2. N	lod	el	Sı	ırf	ac	e	Fl	Lov	y S	er	esc	r	•	•		•	•	•	•	•	24
		8	١.	Pı	e]	Lin	nin	ar	У	Mo	ode	1	•	•	•	•	•	•	•	•	•	•	24
		ì	٠.	Pı	rin	nar	У	Mo	de	21	•	•	•	•	•	•	•	•	•	•	•	•	27
		c		Te	n-	-Ch	an	ne	1	Н	ot-	wi	re	E	r	ре	•	•		•		•	27
III.	EXP	erimen	ITA:	LI	PRO	CE	DU	RE	;	•	•	•	•	•	•	•	•	•	•	•	•	•	32
	A.	DETER GEOME			·	NO.	OF •	` P	RI	MLA •	RY	•	101	ΕI	•			•	•	•	•	•	32
		1. I	re	est	tre	an	ı F	10	W	Cr	ar	ac	te	ri	st	ic	s	•					32
				w.			٠		,	37.	. 7 .		+										22

			ъ.	Fr	equ	enc	y	of	, C	sc	il	.la	ti	or	1	•	•	•	•	•	•	32
			c.	Am	pli	tud	ie	of	, (sc	il	.la	ti	Lor	ı	•	•	•	•	•	•	33
		2.	Bour Pre										n •	or •	ı •	•	•	•	•	•	•	33
		3.	Desi	ign	of	Pr	in	ar	У	Mo	de	1	•	•		•	•	•	•	•	•	34
	в.		SSURI EL .			RIE •	TUE	·IC	N	ON.	•	RI	M.A	RY		•	•	•	•	•	•	35
	c.	BOU	VDARY	L	AYE	R C	HA	RA	CI	ER	IS	TI	CS	5	•	•	•	•		•	•	36
IV.	REST	JLTS		•		•	•	•	•	•	•	•	•	•	•	•	•	•	•	•	•	38
	A .	PREI	LIMI	VAR:	Y M	ODE	EL	•	•	•	•	•	•	•	•	•	•	•	•	•		38
		1.	Osci Fred					-	it •	ud •	e •	٧e	rs	us •	•		•	•	•		•	38
		2.	Bour	nda	ry	Lay	rer	· A	ct	;iv	it	у	•	•	•	•	•		•	•	•	38
	в.	PRIM	MARY	MOI	DEL		•	•	•	•	•	•	•	•	•	•	•	•	•	•	•	40
		1.	Pres	ssu	re	Dis	str	ib	ut	ic	n	•	•	•	•	•	•	•	•	•	•	40
		2.	Bour	nda	гy	Lay	rei	•	•	•	•	•	•	•	•		•	•	•	•	•	48
٧.	CCN	clus	IONS	•		•	•	•	•	•	•		•	•	•		•	•	•	•	•	55
APPENI	XIC	EXP	ERIME	ENT	AL	DAI	'A	•	•	•	•	•	•	•	•	•		•	•	•	•	57
LIST (of Ri	EFERI	ENCES	5			•	•	•	•	•	•	•	•	•	•	•	•	•	•	•	98
INITIA	AL D	ISTR:	IBUT:	ION	LI	ST				•			•		•	•		•	•		•	99

LIST OF TABLES

TABLE	I	Measured Dimensionless Oscillation Amplitude as a Function of Frequency	57
TABLE	II	Experimental Oscillation Frequencies and Amplitudes Data	58
TABLE	III	Measured Surface Pressures 6	; 3
TABLE	IV	Pressure Survey Data 8	36

LIST OF FIGURES

1	Plan View of Wind Tunnel 16
2	Overall Photographic View of the Wind Tunnel
3	Photograph of the Rotating Shutter Valve
4	Preliminary Model 21
5	Typical Tunnel Test Section Velocity
6	Primary Model
7	Primary Model in Test Section
8	Hot-Wire Probe
9	Typical Hot-Wire Anemometer Calibration Curve (Steady Flow) 30
10	Typical Hot-Wire Anemometer Calibration Velocity Profile in Blasius Flow
11	Freestream Oscillation Frequency Versus Amplitude
12	Pressure Distribution 67 Per Cent Blockage
13	Pressure Distribution 67 Per Cent Blockage
14	Pressure Distribution 67 Per Cent Blockage
15	Pressure Distribution 98 Per Cent Blockage
16	Pressure Distribution 98 Per Cent Blockage

17	Typical	Oscillographs	•	•	•	•	•	•	•	•	•	•	٠	•	•	51
18	Typical	Oscillographs	•	•	•	•	•	•	•	•	•	•	•	•	•	52
19	Typical	Oscillographs	•	•	•	•	•	•	•	•	•	•	•	•	•	53
20	Typical	Oscillographs														54

LIST OF SYMBOLS

```
model chord length (inches)
C
          nondimensional coefficient of pressure – C_P = \frac{p - p_0}{\frac{1}{2} \rho \overline{U}^2}
f
          flow oscillation frequency - Hertz (Hz)
          nondimensional flow oscillation amplitude - N_A = \frac{\Delta u}{\tau \tau}
NA
          static pressure (lbf/ft<sup>2</sup>)
p
          freestream ambient pressure (lbf/ft<sup>2</sup>)
Po
          test section dynamic pressure (lbf/ft<sup>2</sup>) q = \frac{1}{2} \rho \overline{U}^2
q
          freestream ambient temperature (OF)
          streamwise component of tunnel velocity (ft/sec)
          velocity fluctuation (ft/sec)
\Delta u
Ū
          mean tunnel velocity (ft/sec)
          streamwise model dimension taken from leading edge
X
X/c
          normalized model location
          dimension normal to flow (inches)
У
          air density (lbm/ft<sup>3</sup>)
          kinematic viscosity (ft<sup>2</sup>/sec)
```

ACKNOWLEDGEMENTS

The author wishes to express his gratitude to Dr. James

A. Miller, Associate Professor of Aeronautics, for his
thoughtful guidance during the present work.

Grateful acknowledgement is also due to the technical engineering staff, and in particular to Messrs. Theodore B. Dunton and Robert A. Besel, for their constant and cheerful cooperation and assistance throughout this study.

I. INTRODUCTION

Increased interest in the design and development of vertical/short takeoff and landing (V/STOL) aircraft has magnified the need to fully understand the complex flow fields associated with their aerodynamic and propulsive systems. The unique capabilities of such vehicles require significant portions of their flight profiles to be performed with much of the aircraft immersed in turbulent flow. When in operation near the ground or landing platform, unsteady, turbulent flow may become the dominant phenomenon.

Given the importance of this area of study and the limitations of the analytical solutions, there is a surprisingly small body of empirical work concerned with unsteady boundary layers caused by freestream flow oscillation. This is perhaps due to a lack of suitable test facilities; that is, facilities that are capable of a wide range of freestream oscillation frequencies and amplitudes. However, significant advances were made by Karlsson Ref. 17 who showed that flow oscillation had little effect on the mean velocity profile of the turbulent boundary layer, and Nickerson Ref. 27 who used hot-wire anemometry in his study of the laminar boundary layer on a flat plate. Despard Ref. 37 studied the separation of a laminar boundary layer using a flow oscillation system developed by Miller Ref. 47. Recently, Telionis Ref. 57

used hot-wire anemometry to investigate the separation and reattachment of boundary layers in unsteady conditions.

The purpose of this investigation was to develop a wind tunnel model useful in determining the effects of various flow oscillation frequencies upon the amplitude of the oscillation and their relationship, if any, to the separation of a turbulent boundary layer. It was also desired to perform initial testing on the equipment associated with a hot-wire anemometry study of the turbulent boundary layer.

II. EXPERIMENTAL EQUIPMENT

A. OSCILLATING FLOW WIND TUNNEL

1. General Description

The low-speed, oscillating flow wind tunnel located in the Aeronautics Laboratories of the Naval Postgraduate School was utilized for this study. This wind tunnel is of the open circuit design, with a 24-inch square by 223-inch long test section. The tunnel inlet is 8-feet square, resulting in a 16:1 contraction ratio. Three high solidity screens located in the inlet section upstream of the nozzle produce measured freestream turbulence intensities of from 0.261 to 0.413 per cent for the test velocities.

The wind tunnel drive consists of two Joy Axivane

Fans in series, each of which has an internal 100 horsepower,

direct connected, 1750 rpm motor. The fan blades are

internally adjustable through a pitch range of 25 to 55

degrees, providing a wide operating range. Each fan has a

separate set of variable inlet vanes that are multi-leaf in

design and are remotely adjustable to afford fine control of

test section mean velocity during tunnel operation. The

tunnel velocity range is from 10 to 250 feet per second,

although the maximum mean velocity used for this study was

148 feet per second. The inlet vanes are set to preswirl

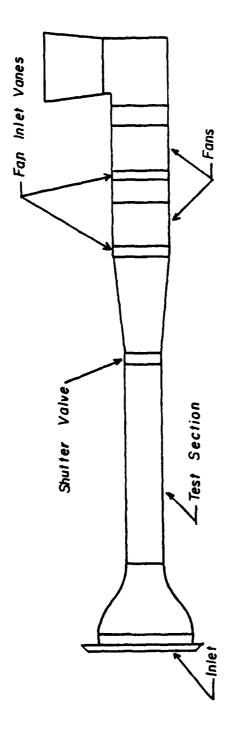
the air flow in the direction of fan rotation in order to

reduce fanloads. To minimize wall deflections caused by large and almost instantaneous changes in static pressure, the test section upper and lower walls are constructed of continuous pieces of two-inch thick aluminum, 24 inches wide and 223 inches in length. The test section wall facing the tunnel control console is composed of three hinged, two-inch thick, stress relieved Lucite panels. The Lucite doors, while normally secured in the closed position by twelve large bolts per panel during tunnel operation, may be hydraulically raised for access to the test section. The back wall of the test section is also composed of three panels that are manually removable in order to facilitate model installation. For this experimental study, the upstream back wall panel was made of Lucite and the two downstream panels were constructed of two-inch thick plywood to allow for instrumentation installation.

A plan view of the oscillating flow wind tunnel is shown in Figure 1. An overall photographic view of the tunnel as seen from downstream of the fan inlet vanes is shown in Figure 2.

2. Flow Oscillation System

A sinusoidal velocity component is introduced into the mean freestream flow by harmonic solid blockage variations downstream of the test section. This is accomplished by the use of four horizontal, rotating shutter blades that completely span the trailing edge of the test section. Four steel shafts,



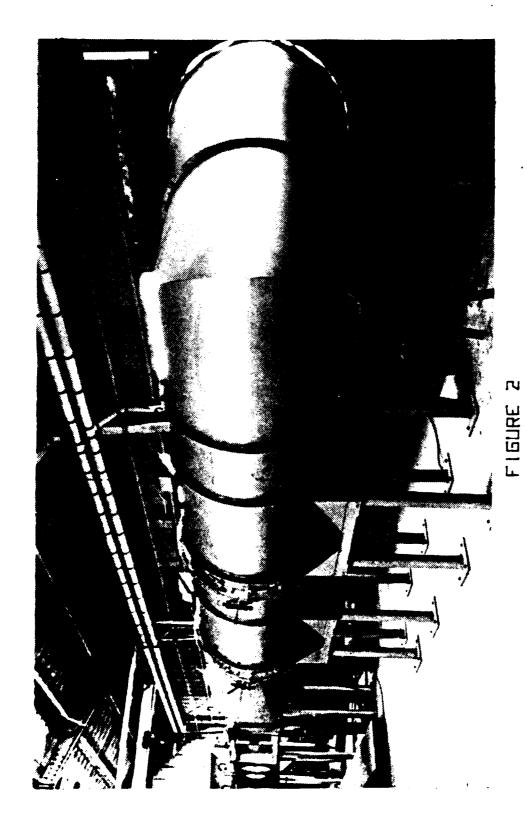
PLAN VIEW OF WIND TUNNEL

FIGURE 1

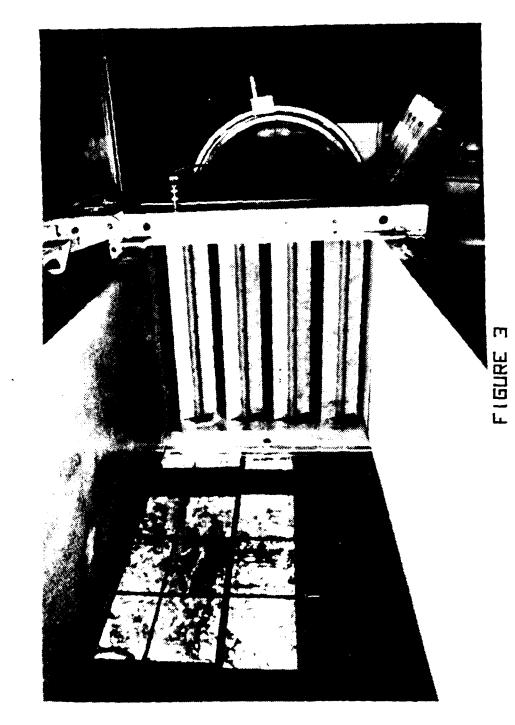
į

equidistant from each other and the test section walls, are slotted to accept various width flat blades, thereby forming a variable sized, multi-slotted, butterfly-type valve. A photographic view of the shutters looking downstream from the test section is shown in Figure 3. The use of this type of shutter system to produce flow oscillations through a large range of frequencies and amplitudes was employed by Karlsson $\sqrt{\text{Ref. 17}}$, and is identical to that employed by Miller $\sqrt{\text{Ref. }}$. The drive for the shutter system is a five horsepower variable speed electric motor coupled to the bottom shaft of the shutter system via a belt and pulley system in order to produce a wide variety of frequencies. The upper three shutter shafts are connected to each other and the driven shaft by timing belts to insure that all four shutters rotate in phase. The total range of remotely selectable shutter frequencies is from 0.1 to 240 Hertz. This investigation employed frequencies of from 1 to 100 Hertz.

Gross oscillation amplitude may be changed by the installation of one of several sets of shutter blades having different widths and therefore different blockage ratios. The range of test section blockage produced by the various shutter blades was from 25 to 98 per cent. This investigation primarily utilized test section blockage of 67 and 98 per cent, resulting in amplitude variations of from 3 to 108 per cent of the mean freestream velocity. The shutters fixed in the fully open position cause a non-oscillatory blockage of approximately five per cent.



DVERFILL PHOTOGRAPHIC VIEW OF THE WIND TUNNEL



PHOTOGRAPH OF THE ROTATING SHUTTER VALVE.

19

B. MODELS

1. Preliminary Models

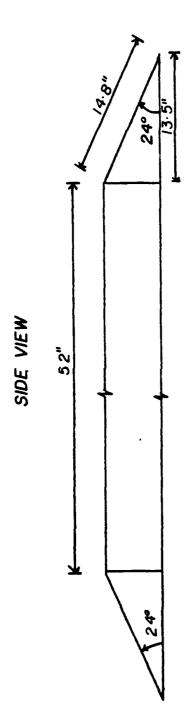
In order to effectively investigate the effects of unsteady flow on turbulent boundary layer separation, it was necessary to develop a model that would produce boundary layer separation in the neighborhood of a location in which the instrumentation could be conveniently introduced. In order to accomplish this, several possible model geometries were inexpensively constructed and tested in order to evolve a configuration to be employed on a more fully instrumented, and rugged, primary model.

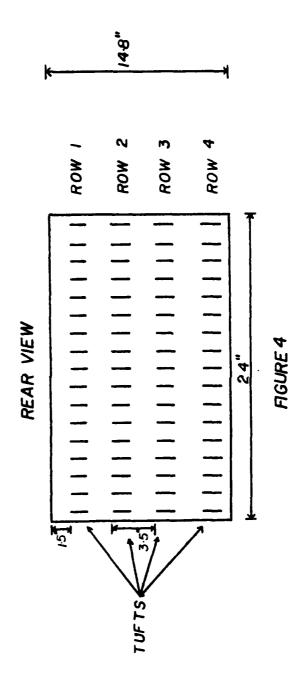
The preliminary models were of all wood construction and were two-dimensional shapes spanning the width of the test section, mounted to the floor of the test section, causing convergence and divergence of the section. The models all used a 54-inch long 6-inch thick main body with provisions for interchangeable leading and trailing edge sections. Several leading/trailing edge sections were constructed with inclines of 12, 18, and 24 degrees. Figure 4 is a sketch of a preliminary model showing the main body with 24 degree leading and trailing edge sections installed. During the preliminary study, the trailing edge section was tufted for visual indications of turbulent flow and boundary layer separation.

2. Primary Model

Based on the results of the preliminary investigation, a primary model was designed.

PRELIMINARY MODEL

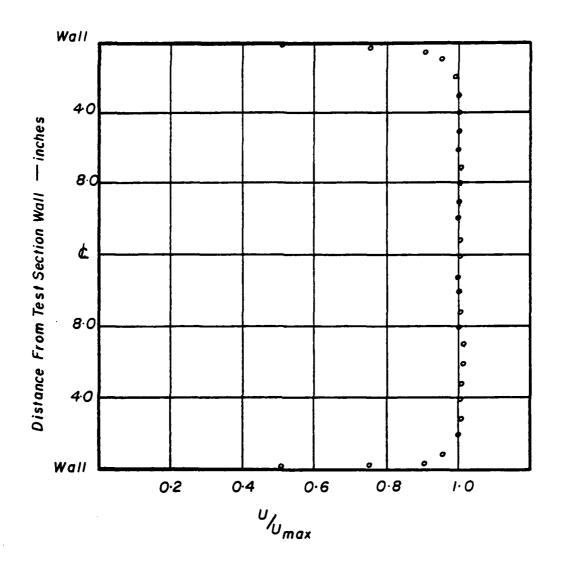




The primary model was constructed of a continuous upper surface skin of 0.100-inch thick aluminum, 62.5 inches long, which was welded to three continuous pieces of 0.25-inch thick aluminum forming three bulkheads, which were then welded to a single, flat 0.25-inch aluminum deck. The primary model was of the same general shape as the preliminary model, with an 18 degree inclined trailing edge and parabolic leading edge. Ports were cut into the six-inch high bulkheads to facilitate service of surface pressure instrumentation and to afford access to the bolts mounting the model to the tunnel floor.

Along the port side, looking forward, twenty-seven .040-inch static pressure ports were positioned four inches from the test section wall. These pressure ports ran from 22 inches aft of the leading edge in two-inch intervals until just forward of the point of the after-body ramp, where they were positioned each inch, terminating 2.5 inches from the trailing edge. The lateral positioning of these ports was determined from measured velocity profiles shown in Figure $5\sqrt{Ref}$. 27 in order to be outside the wall boundary layer and at the same time to leave the centerline of the model free of ports and avoid interference with the multichannel hot-wire probe.

Two, two-inch wide, 25.5-inch long strips of 0.040-inch thick steel were imbedded into the model skin to enable a magnetically mounted multi-channel, hot-wire anemometer



TYPICAL TUNNEL TEST SECTION VELOCITY PROFILE

(U_{max} = 20 ft/sec)

FIGURE 5

probe to be easily positioned for boundary layer surveys. Figure 6 shows the relative positions of the steel tracks and the pressure ports.

Figure 7 shows the primary model in position in the test section of the wind tunnel with the instrumentation installed.

C. INSTRUMENTATION

1. Freestream Flow Sensors

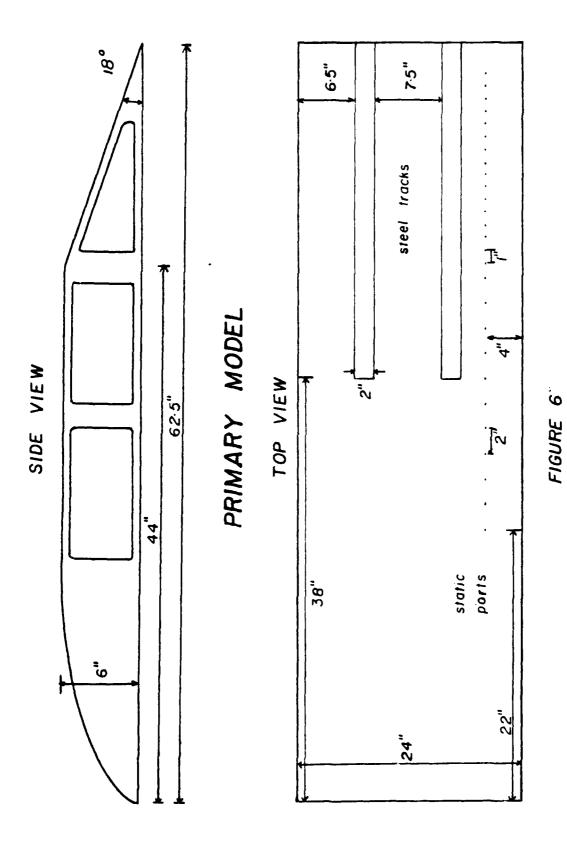
A standard pitot-static tube and a hot-wire anemometer probe were located six inches above the model at mid-chord of both the preliminary and primary models to determine mean freestream velocity. Dynamic pressure was read from a Meriam micro-manometer for the preliminary model, and from a 50-tube water manometer used also to measure the surface pressure distribution for the primary model. The freestream turbulence was measured with a linearized hot-wire anemometer described in Ref. 3.

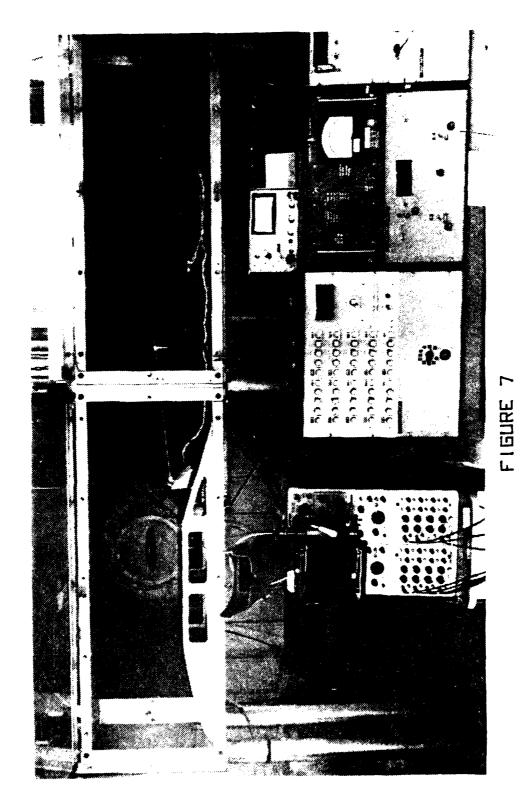
The frequency of the shutter rotation, and therefore, the frequency of flow oscillation, was measured with a digital counter which read an electrical signal developed by an optical system employing a stationary point light source and a rotating chopper wheel fixed to the top shutter valve shaft.

2. Model Surface Flow Sensors

a. Preliminary Model

The principal method of investigation for the preliminary model was the observation of tufts attached to





PRIMARY MODEL IN THE TEST SECTION

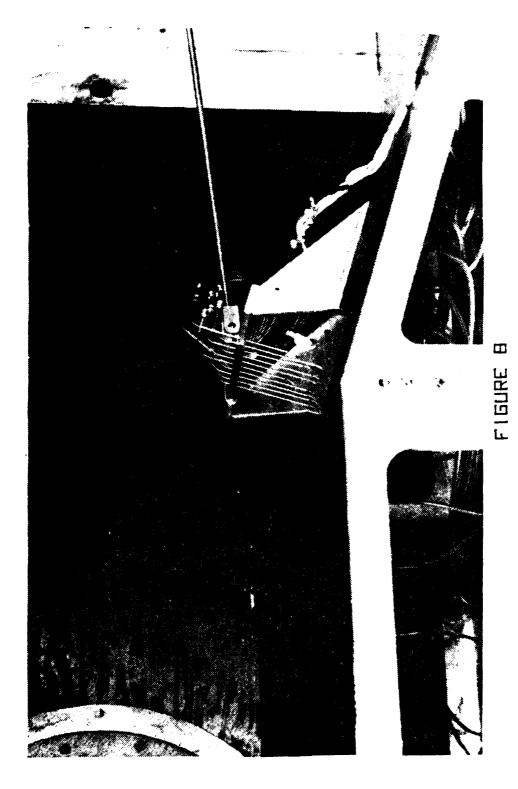
the divergent section. In order to facilitate these observations, a stroboscopic light was electrically triggered by a contact on the uppermost shutter shaft to permit an optical freezing of the motion of the tufts in the oscillating flow. The stroboscopic light trigger was mounted on a rotatable base assembly which allowed the tuft action to be viewed at any phase of the shutter cycle.

b. Primary Model

The surface pressure distribution over the aft two-thirds of the primary model was measured by 27 static pressure ports on the model surface. The first 11 of these ports were located on the constant area section of the model, with the remaining 16 ports located on the diverging section. Pressures at these ports were read, along with the freestream dynamic pressure, on a 50-tube water manometer calibrated in centimeters.

c. Ten-Channel Hot-Wire Probe

In order to investigate the boundary layer along the surface of the primary model, an aluminum carriage supporting ten constant temperature hot-wire anemometer probes was employed. Each hot-wire could be individually positioned from the surface to approximately three inches above the surface. The carriage stood on magnetic feet that matched the steel tracks in the model. This allowed a continuous chordwise traverse of the probe from six inches upstream of the point of divergence to approximately two inches



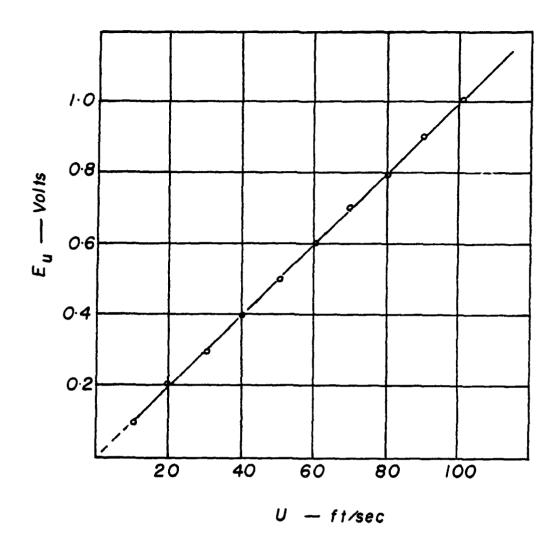
HOT-WIRE PROBE

from the trailing edge. During oscillating flow tunnel operation, the probe exhibited a tendency to "walk" along its magnetic track. In order to overcome this, a simple support rig was installed consisting of a single threaded rod, mounted to the hot-wire carriage and a streamlined stand, anchored to the tunnel floor aft of the model. Figure 8 shows the hot-wire probe in position of the model.

The ten-channel hot-wire circuits were identical to the one employed in the freestream hot-wire. Figure 9 is a typical calibration curve for non-oscillating flow, and Figure 10, a typical calibration in Blasius flow, as demonstrated by Despard \sqrt{Ref} . 37 and Allen \sqrt{Ref} . 67.

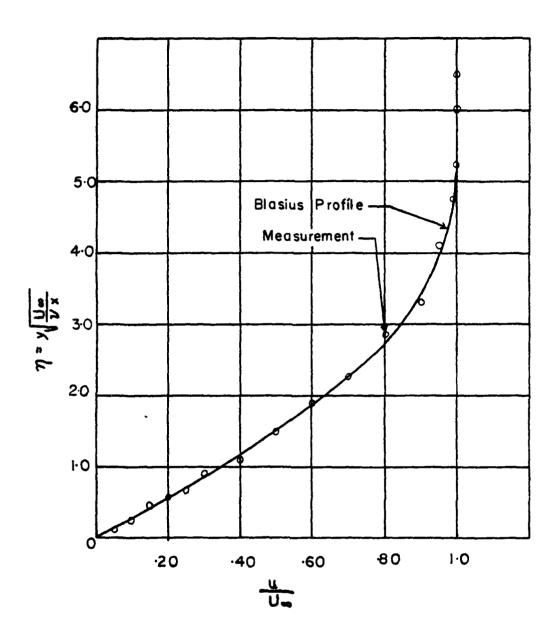
Each sensing element was a 0.00015-inch diameter tungsten filament one-eighth-inch long copper plated at its ends to facilitate mounting. The effective sensing length of each wire was approximately 0.080 inches.

Signals representing total instantaneous velocities were produced at the outputs due to the DC coupled circuitry of the anemometers. The oscillating velocity components of the total velocity are proportional to the alternating current component of the anemometer output, and were displayed on a Tektronix 555 Dual Beam oscilloscope. The oscilloscope was equipped with two, four-channel preamplifiers, permitting a maximum of eight hot-wire outputs to be viewed simultaneously. A Tektronix oscilloscope camera was used to record the multichannel display.



TYPICAL HOT WIRE ANEMOMETER CALIBRATION CURVE (STEADY FLOW)

FIGURE 9



TYPICAL HOT WIRE ANEMOMETER CALIBRATION
VELOCITY PROFILE IN BLASIUS FLOW

III. EXPERIMENTAL PROCEDURE

A. DETERMINATION OF PRIMARY MODEL GEOMETRY

1. Freestream Flow Characteristics

Prior to investigating the boundary layer characteristics of the preliminary model, it was necessary to determine the operating ranges of the freestream flow variables in oscillating flow with the model installed in the test section. The flow variables of interest were: mean tunnel velocity, frequency of oscillation, and amplitude of oscillation.

a. Mean Tunnel Velocity

In order to insure a turbulent boundary layer over the trailing edge of the model, mean tunnel velocities of 111 feet per second, 132.37 feet per second, and 148.16 feet per second, were set and maintained via the variable inlet vanes and measured with the pitot tube. These velocities yielded Reynolds numbers, under average ambient conditions of 3.8×10^6 , 4.5×10^6 , and 5.1×10^6 for a characteristic length of 70 inches.

b. Frequency of Oscillation

Shutter rotation frequencies, and therefore, flow oscillation frequencies, of from 1 to 100 Hertz were investigated in the initial testing. The frequencies were measured by the previously described electro-optical system and were read directly from the digital frequency counter. Shutter frequency

was varied from one to six Hertz in one Hertz increments, and from 20 to 100 Hertz in two Hertz increments for a single mean velocity. The frequency range of from 6 to 20 Hertz was not investigated due to shutter drive gearing difficulties. Oscillation frequency was then retarded from 100 Hertz to one Hertz by reversing the above procedure while maintaining a constant mean tunnel velocity.

c. Amplitude of Oscillation

The amplitude of oscillation was measured with the single-channel freestream hot-wire anemometer and the AC component was read with a Ballantine true RMS voltmeter. These readings were of change in streamwise velocity normalized with freestream mean velocity $(N_{\rm A})$. Amplitude readings were taken at each frequency while traversing the frequency range in the upward direction, then confirmed while descending the frequency range.

Several experimental runs were made at each of the mean velocities with two full data collection runs made for each of the lower mean velocities, and one for the highest velocity. Due to instrument fluctuations experienced at frequencies less than four Hertz, the oscillation amplitude values for the lowest frequencies were considered at best approximate and were not reported.

2. Boundary Layer Separation on Preliminary Model

The paramount purpose of the preliminary investigation was to determine the combination of leading and trailing edge

sections that would give rise to turbulent boundary layer separation on the trailing edge section. The use of tufts on the trailing edge sections was believed to be the most reliable and expedient method to study the desired phenomenon, given the lack of instrumentation incorporated into the preliminary model.

It became clear early in the testing that the leading edge section had no real effect upon the desired conditions, therefore, the 24-degree leading edge section was fixed in position for the duration of the experiment.

The trailing edge sections were affixed with tufts, as shown in Figure 4, and the model was run through the range of freestream conditions. The stroboscopic light was connected to the shutter shaft and cam, and was positioned so as to effectively light the tufts. The reaction of the tufts to the flow was visually studied to determine the effects of flow oscillation on boundary layer separation. Numerous experimental runs were made with the several trailing edge sections under the various freestream conditions and with stroboscopic and normal lighting.

3. Design of Primary Model

From the results of the preliminary investigation, it was concluded that the primary model should be of similar dimensions to those of the preliminary model with a trailing edge incline of 18 degrees to best produce a turbulent boundary layer that would separate at the desired location

under test conditions. Due to structural difficulties experienced with the wooden preliminary models, it was also concluded that the primary model should be designed with increased strength. The primary model constructed under the above criteria was somewhat smaller in the chordwise dimension than the preliminary model. The shorter chordwise dimension positioned the divergent section of the model 45 inches from the leading edge. This shortening of the chord for the primary model allowed a single test section panel bolting and unbolting requirement for maintenance ease, but still produced a sufficiently high Reynolds number at one-half of the lowest preliminary test mean velocity. Jacobs Ref. 7 determined that a turbulent boundary layer would occur at a Reynolds number of approximately 1.0 x 106 for a flat plate, in this identical wind tunnel under similar test conditions. The primary model would experience a Reynolds number of 1.2 \times 10⁶ at the point of divergence with a mean velocity of 55 feet per second.

B. PRESSURE DISTRIBUTION ON PRIMARY MODEL

The pressure distribution over the primary model from a position 22 inches aft of the leading edge to the trailing edge was determined by use of static pressure ports and a manometer board. Several runs were made with both the 67 per cent and 98 per cent occlusion shutter blades installed. The frequency values used for data collection were 1, 2, 4, 6, and 25 Herts, for both sets of shutter blades, 30 to 100 Hertz,

and 30 to 70 Hertz, in ten Hertz increments, for the 67 per cent and 98 per cent blades respectively. Considering that the nondimensional coefficient of pressure (C_p) was to be the ultimate output of these runs, mean tunnel velocity was not necessarily kept constant throughout the frequency range.

A maximum freestream oscillation amplitude at six Hertz was noted during this phase of the experiment as being 108 per cent, while using the large shutter blades.

C. BOUNDARY LAYER CHARACTERISTICS

The boundary layer over the model was monitored to ensure turbulent flow with the aide of the ten-channel hot-wire probe and the eight-trace oscilloscope. Evidence of turbulent flow was taken to be the characteristic oscilloscope trace for turbulence as discussed by Bradshaw $\sqrt{Ref. 87}$.

Prior to each experimental run, calibration of the hotwire anemometers was carried out in situ. The positioning
of the boundary layer probe's wires with respect to height
above the model was then set. The final setting of the
individual anemometers was accomplished after the calibration
in consideration of the possibility that one or more of the
wires may have failed the calibration procedure. The wire
heights could then be adjusted so as to provide adequate
coverage of the boundary layer without the need to remove the
entire probe for repair. For the purposes of this initial
investigation, a meaningful run could be made with the freestream anemometer and five of the boundary layer anemometers

being in calibration. The heights above the model surface, for this worst-case hot-wire availability, were a wire each at 0.05, 0.10, 0.20, 0.30, and 0.40 inches.

The wind tunnel was set to operate with an oscillation frequency of 20 Hertz, oscillation amplitude of 18 per cent, mean velocity of 15 feet per second, and the large shutter blades installed. Photographs of the multi-trace oscilloscope were taken with the hot-wire carriage set at various chordwise positions. After completing this overall chorwise survey, the carriage was positioned five inches downstream of the point of model divergence. This mean freestream velocity was then varied from 22 to 88 feet per second, and the oscillation frequency varied from 20 to 70 Hertz, to complete this initial turbulent boundary layer survey.

IV. RESULTS

A. PRELIMINARY MODEL

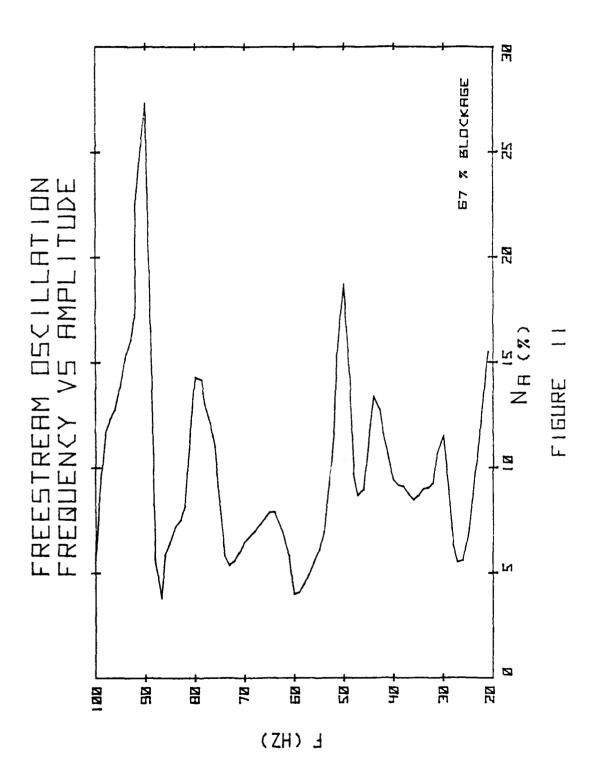
1. Oscillation Amplitude Versus Frequency

The data collected from the freestream amplitude investigation for the selected mean velocities indicates that the mean velocity had little or no effect upon the nondimensional amplitude factor, N_A . A study of the data shows that for a specific oscillation frequency (f), the values obtained for N_A were usually within one per cent of each other for any of the mean velocities tested. Due to this similarlity, the values of amplitudes, for a given frequency, were arithmetically averaged in order to be plotted against oscillation frequency. Figure 11 is the graph of oscillation amplitude versus oscillation frequency for a tunnel occlusion of 67 per cent.

Figure 11 reveals the bulk of the values of N_A to lie between 5 and 15 per cent. Values larger than 15 per cent are seen to have occurred at 21 - 22 Hz, and 90 - 94 Hz. Other, less prominent, peaks occurred in the vicinity of 30 Hz, 40 Hz, and 80 Hz.

2. Boundary Laver Activity

The tufting of the preliminary model, as shown in Figure 4, clearly evidenced a turbulent boundary layer in existence over the area of interest in the case of the 18 and



24 degree diverging model sections. Close study of the 24 degree section showed that the boundary layer remained attached to the Row 1 area (Figure 4) as oscillation began from steady flow conditions. The boundary layer remained attached in this area until approximately 28 Hz when the entire diverging section became stalled. The fully stalled condition continued through approximately 70 Hz, when the Row 1 area again showed turbulent boundary layer attachment. The entire section again became fully stalled at the 80 Hz reading and remained so through the test limit of 100 Hz.

The 18 degree diverging section displayed similar frequency response but the turbulent boundary layer attachment point had moved into the Row 2 and 3 area.

These results were quite repeatable and consistent throughout the range of mean velocities (111 - 148 ft/sec).

B. PRIMARY MODEL

1. Pressure Distribution

A surface pressure survey was conducted with both the 67 per cent and the 98 per cent flow blockage shutter blades installed. The range of oscillation frequency (f) was 1 Hz to 100 Hz for the former and 1 Hz to 70 Hz for the latter shutter blade configuration. To preclude physical damage, the larger occlusion tests were limited to the 70 Hz level due to violent tunnel behavior above this value. In fact, data was collected at 80 Hz with the 98 per cent blades installed,

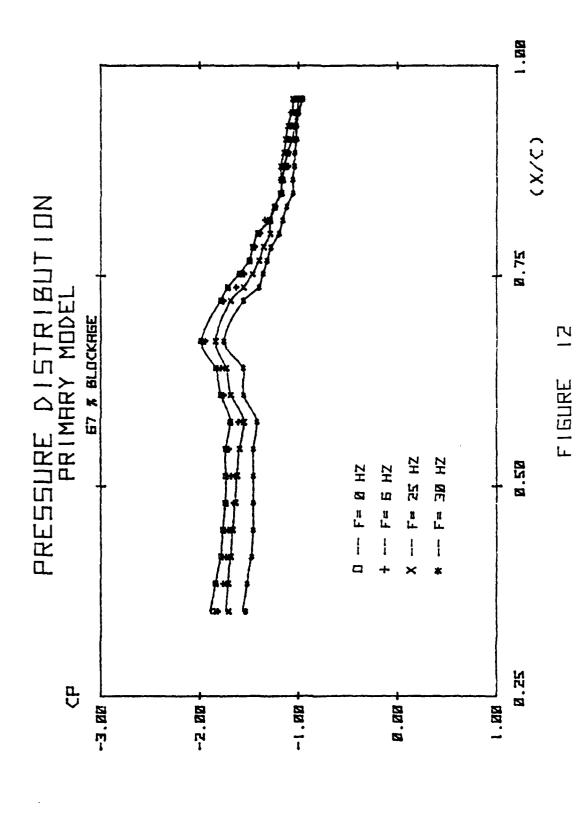
but, due to equipment vibration, that data point was not repeated and therefore considered unreliable.

Figures 12, 13, and 14 depict the pressure coefficient (C_p) measured at each of the 27 pressure ports from 31.2 to 96 per cent chord for the case of 67 per cent flow blockage. The peak of these curves (lowest value of C_p) occurs uniformly at port number 11 which was located just upstream of the point of model divergence at 67 per cent chord. As may be seen, the C_p curves remain generally smooth throughout the frequency range with all values of C_p remaining more positive than the steady state (f = 0 Hz) values.

The curves for frequencies of one Hz, two Hz, and four Hz were identical to the curve for six Hz and were not presented. As may be seen, the curve for six Hz is nearly identical to the steady state value.

The values for 25 Hz, 40 Hz, and 80 Hz are also nearly identical to each other. The curves of 30 Hz, 50 Hz, and 90 Hz duplicate each other and are somewhat less negative than the 25 Hz series. The values for 100 Hz stand alone at the most positive edge of the family.

The differences between the curves becomes generally less pronounced downstream of the point of model divergence. The values for 6 Hz, 25 Hz, and 30 Hz all merge at the 90 per cent chord point. The 50 Hz and 60 Hz curves become nearly coincident aft of 70 per cent chord. A similar situation occurs for the 90 and 100 Hz C_p values.



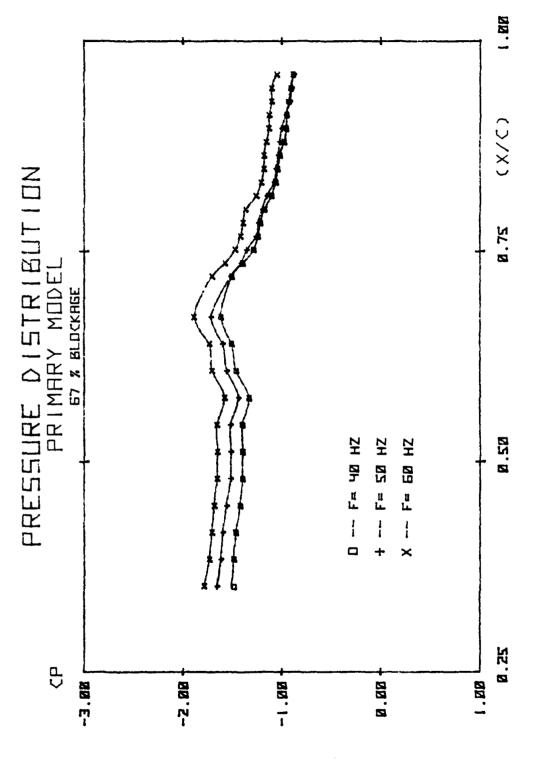
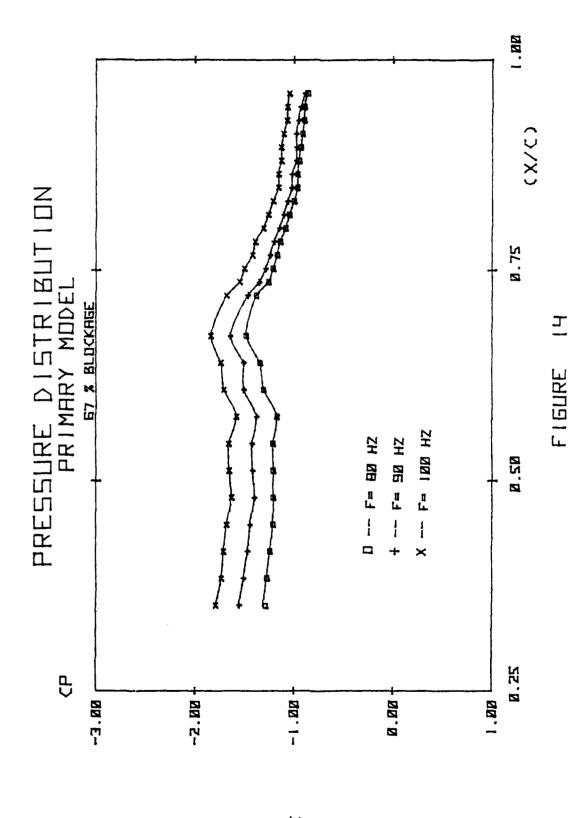


FIGURE 13.



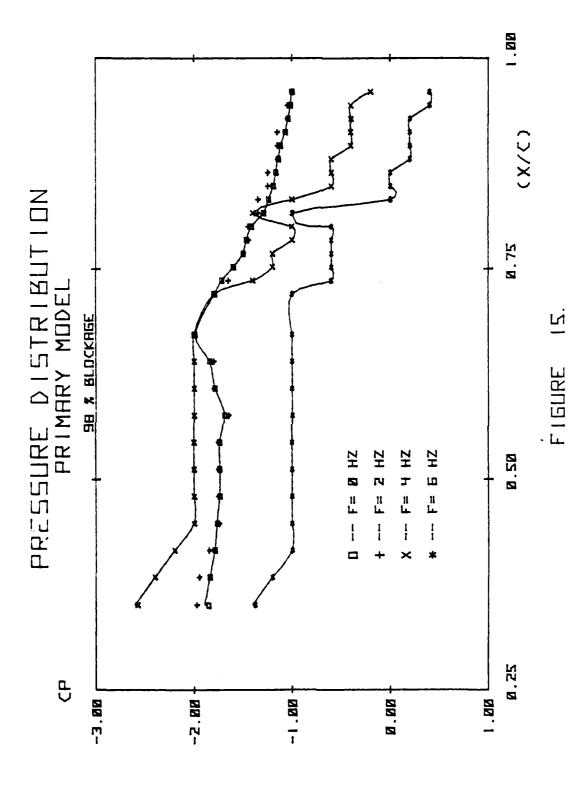
Figures 15 and 16 present the pressure coefficient versus chord for the 98 per cent occlusion shutter blades. Here the one Hz and two Hz values track very closely with the steady flow $C_{\rm p}$ curve.

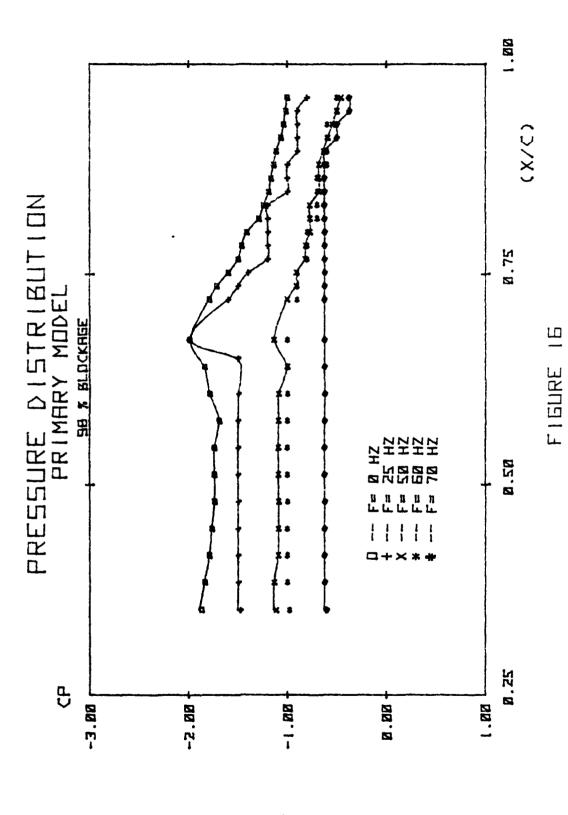
This family of curves behaves much differently than the 67 per cent blockage curves. In general, there is a much larger range of values of $C_{\rm p}$ over the frequency range than the smaller blockage, and the curves tend to be much more erratic in their behavior. Except as noted, no two curves are alike.

The curve depicting an f of four Hz begins at values much less negative than the steady state case, joining it at the point of model divergence and then dropping sharply off until the 80 per cent chord point where it again rises to become more negative than steady state only to immediately fall off to a C_p value of nearly zero.

The six Hz curve is nearly identical to the shape of the four Hz curve with the exception that the $C_{\rm p}$ value remains constant until approximately 70 per cent chord. The six Hz curve also fluctuates at 80 per cent chord then drops off sharply to $C_{\rm p}$ values near +0.5.

As the oscillation frequency is increased, the curves become less erratic in their behavior. The 25 Hz curve shows some of the tendencies of the 4 and 6 Hz curves, but at a reduced level. The 60 Hz curve is nearly flat, while the 70 Hz curve is indeed a straight line until approximately 90 per cent chord, where it slowly increases.





It should be noted that the large water manometer board was well suited for this investigation due to the natural damping of its measurements. It was relatively simple to obtain the pressure data without the averaging necessary with a faster response system, especially at the higher frequencies.

2. Boundary Layer

The turbulent boundary layer investigation performed on the primary model was designed as a preliminary check on apparatus suitability, operating procedures, and identification of potential problems for future study of the flow phenomenon. With these goals in mind, several series of experimental runs were performed with the ten-channel hot-wire probe, and all associated equipment in position and operating.

It was found that the setup and electronic calibration of the probe could be performed efficiently and accurately after a short initial learning period. This included the manual setting of individual anemometer heights and identification of any broken or questionable hot-wires.

The security of the probe on its magnetic feet and steel traverse was improved considerably by the addition of the supporting rod and stand. The probe was easy to move to any position and was secure in that position throughout the frequency range of from 20 Hz to 70 Hz at several mean velocities.

The satisfactory performance of the primary model and its measuring equipment was demonstrated during a

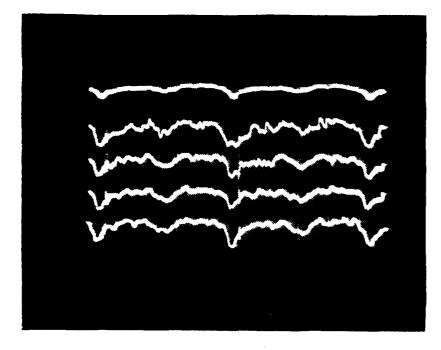
simulated data collection run. Figures 17 - 20 are typical of the oscillographs taken from the multi-trace oscilloscope during this final experiment. The uppermost trace depicts the freestream flow read from the test section single hot-wire anemometer. The second through fifth trace shows the boundary layer flow at 0.05 inches, 0.10 inches, 0.20 inches, and 0.30 inches above the model surface. The sixth trace, if present, depicts the boundary layer at 0.40 inches above the model. The tunnel was run with the 98 per cent occlusion shutter blades installed.

The freestream conditions for Figures 17 and 18 were mean velocity set at 42 feet per second, oscillation frequency set at 20 Hz, yielding an oscillation amplitude of 18 per cent. The variable in the four oscillographs is the location of the probe on the model. The range of locations shown are from 2 inches upstream of the model divergence point, $(^{x}/c = .67)$, to 14 inches downstream, $(^{x}/c = .93)$.

The oscillographs in Figure 19 and the upper one in Figure 20 are for a fixed probe location of five inches downstream of the model divergence point, $(^{X}/c = .78)$. Mean tunnel velocity was increased to 51.4 feet per second and frequency was made the variable. Note the increase in oscillation amplitude for the 20 Hz case. It was shown for the 67 per cent occlusion blades that the mean freestream velocity had no effect upon the value of N_{A} . The lower oscillograph in Figure 20 is at the same conditions as the 20 Hz

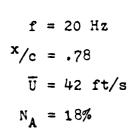
oscillograph of Figure 19 with the exception that the mean velocity was increased to 103 feet per second. The oscillation amplitude, however, remained the same as the 51.4 feet per second case.

It is clear that the boundary layer was entirely turbulent throughout this test.



f = 20 Hz x/c = .67 $\overline{U} = 42 \text{ ft/s}$ $N_A = 18\%$

TYPICAL CSCILLOGRAPHS



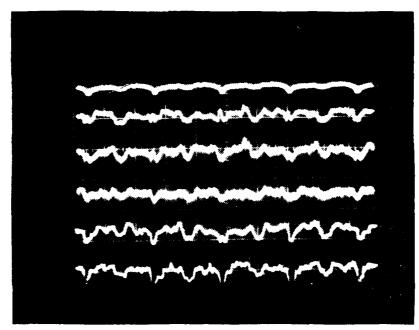
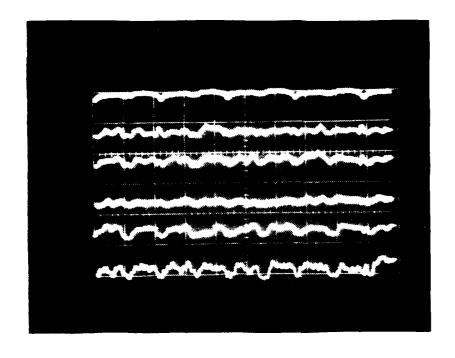


FIGURE 17



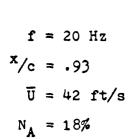
f = 20 Hz

$$x/c = .84$$

$$\overline{U} = 42 \text{ ft/s}$$

$$N_{\mathbf{A}} = 18\%$$

TYPICAL OSCILLOGRAPHS



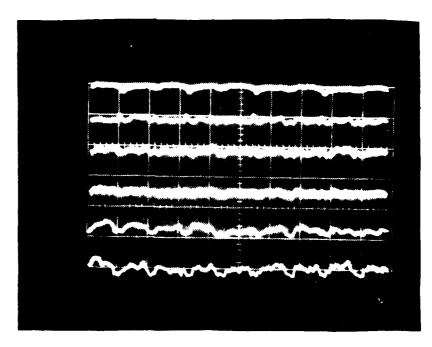
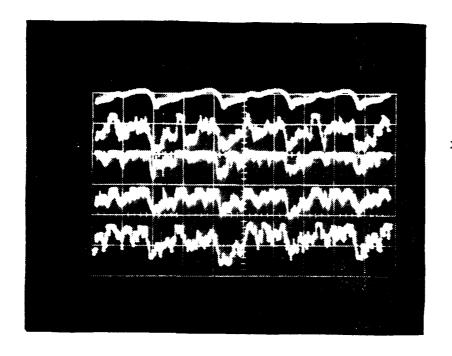
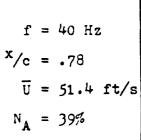


FIGURE 18



f = 20 Hz x/c = .78 $\overline{u} = 51.4 \text{ ft/s}$ $N_A = 35\%$

TYPICAL OSCILLOGRAPHS



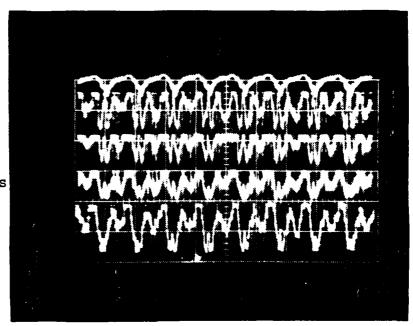
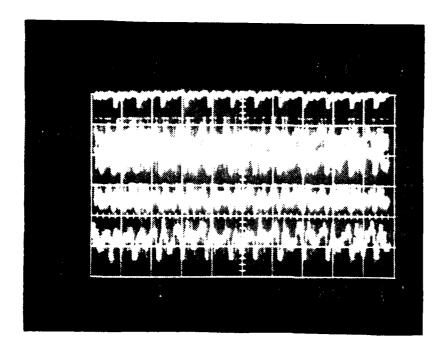


FIGURE 19.



f = 20 Hz

x/c = .77

 $N_{A} = 35\%$

$$f = 60 \text{ Hz}$$
 $x/c = .78$
 $\overline{U} = 51.4 \text{ ft/s}$
 $N_A = 29\%$

TYPICAL OSCILLOGRAPHS

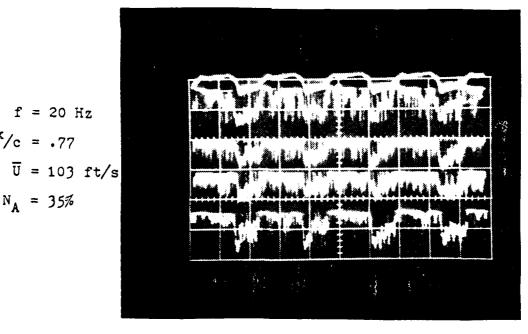


FIGURE 20

V. CONCLUSIONS

From the results described above, the following conclusions may be drawn:

Oscillation amplitude is clearly a function of oscillation frequency in this tunnel and may increase dramatically with little change in frequency. The amplitude change of from less than 5 per cent at 88 Hz to over 27 per cent at 90 Hz is certain evidence of this fact.

The effect of mean freestream velocity upon oscillation amplitude, when coupled with large flow occlusions, is not clear. The mean freestream velocity showed to have no noticeable effect upon amplitude in the 67 per cent flow occlusion study; however, the 98 per cent occlusion tests showed mixed results. The amplitude doubled with a 22 per cent increase in tunnel velocity at one point, but then remained constant with a 100 per cent increase in mean velocity.

Turbulent boundary layer separation over a two-dimensional body may be affected by flow oscillation frequency. The results of the tuft experiment support this conclusion.

The primary model design and construction is satisfactory for the investigation of the effect of freestream oscillation on turbulent boundary layer separation.

The ten-channel hot-wire boundary layer probe performed well, but due to the large aerodynamic forces capable of being produced by the tunnel, should be adequately braced in the streamwise direction.

APPENDIX

EXPERIMENTAL DATA

TABLE I

MEASURED DIMENSIONLESS OSCILLATION AMPLITUDE AS A FUNCTION OF FREQUENCY

f	N _A	f	NA
(HZ)	N (%)	(HZ)	(%)
4561468024680246802468 222233333344446802468	18.0 11.5.5 11.5.8 11.5.8 11.5.8 11.5.9 11.5.9 11.5.9 11.7.9 11.7.9 11.7.9 11.7.9 11.7.9 11.7.9 11.7.9	60 62 64 66 68 70 72 74 76 80 82 84 88 90 92 94 98 100	4.8 96 0.5 6 8 0 0 13.2 2 96 3 3 2 8 6 5 11.5 5 7 7 152 8 6 5
) 0	7.7		

NOTE: 1. Data reduced from raw data by arithmetic

averaging.

f is Oscillation Frequency in Hertz.

Na is nondimensional Oscillation Amplitude

(\(\subseteq \overline{\pi} \)).

TABLE II

q = 7 cr	n H ₂ 0	p = 29.98	in Hg	$T = 70^{\circ}F$
f (HZ)	N _A (%)		f (H2)	N _A (%)
4562468024680246802468 22223333334444555555	17.5 15.0 15.0 15.0 15.0 15.0 16.5 10.0 11.0 18.0 11.0 18.0 11.0 18.0 11.0 18.0 11.0 18.0 11.0 18.0 11.0 18.0 11.0 18.0 11.0 18.0 19.0 19.0 19.0 19.0 19.0 19.0 19.0 19		60 62 66 68 68 77 76 80 88 88 99 98 98 100	5.0 6.5 8.0 7.0 6.0 13.0 16.0 16.0 17.0 17.0 17.0 17.0 17.0 17.0 17.0

q = 7	cm H ₂ 0	p = 29.98 in Hg	$T = 70^{\circ}F$
f (HZ)	N _A (%)	f (H2)	N _A (%)
45614680246807680468 222233333444455555	18.0 15.5 15.5 19.0 18.5 19.0 18.5 19.0 18.5 19.0 18.5 19.0 18.5 19.0 18.5 19.0 19.0 19.0 19.0 19.0 19.0 19.0 19.0	60 62 64 668 70 77 77 78 82 88 88 99 99 98	5.50 5.50
38	4.5	100	7.0

q = 10	cm H ₂ O	p = 30.00 in Hg	T = 68°F
f	N _A	f	N _A
(HZ)	(%)	(HZ)	(%)
134680246804468024680 2222233333344468024680	14.0 13.0 7.0 10.0 11.0 99.5 14.0 11.0 11.0 11.0 11.0 11.0 11.0	62 64 66 68 70 77 76 88 88 88 99 99 99 98	4.550852085050000000000000000000000000000
60	3.5	100	5.0

q = 10 c	m H ₂ 0	p = 30.00	in Hg	$T = 68^{\circ}F$
f (HZ)	N _A (%)		f (HZ)	^N A (%)
24680246804680246802 33333344680246802	14.5 96.5 7.0 12.5 99.5 99.5 10.0 10.5 11.8 6.0 11.8 6.5 5.5 5.5	•	6680246802468099680 100000000000000000000000000000000000	7.0 7.0 7.0 6.0 10.5

q = 12.	5 cm H ₂ 0	p = 29.99 in Hg	$T = 69^{\circ}F$
f (HZ)	^N A (≉)	f (HZ)	N _A (%)
4 56 146802460468046 2222333344468555	19.0 14.5 12.0 16.0 8.5 6.0 11.5 9.0 8.0 9.0 13.0 9.5 20.0	58 60 64 668 70 74 76 82 84 88 90 92 94 100	3.5 3.0 8.5 7.6.0 6.0 10.0 5.0 13.0 13.0
56	6.5 5.0	100	13.0 4.5

TABLE III

	2024.14192	2026.16752	2027.21032	2028.23312	2029,25592	2631.30152	2632.32432	2033,34712	2033.9608
	PORT # 19 X/C= 0.632 PORT PRESS= CP= -1.24	PORT # 20 X:C= 0.848 PORT FFESS= CP= -1.19	PORT # 21 X/C= 0.864 PORT PRESS= CP= -1.165	POPT # 22 X/C= 0.88 POPT PFESS= CF= -1.14	PORT # 23 X/C= 0.896 PORT PRESS= CP= -1.115	PORT # 24 X*C= 0.912 PORT PRESS= CP= -1.065	FORT # 25 X.C= 0.928 PORT RFESS= CP= -1.04	PORT # 26 X/C= 0.944 PORT PRESS= CP= -1.015	PORT # 27 X/C# 0.96 PORT PRESS# GP# -1
	1999.59472	1993,45792	2001.64032	2664.78872	2009.4136	2068.736	2014.93672	2016.98232	2022.09632
	PORT # 18 X/C= 4.64 PORT PRESS= 1 CP= -1.84	PORT # 11 X/C= 0.672 PORT PRESS= CP= -1.99	PORT # 12 X/C= 0.72 PORT PRESS= CP= -1.79	POPT # 13 X.C. 0.736 POPT PRESS= CP= -1.715	PORT # 14 X/C= 0.752 PORT PRESS= CP= -1.6	PORT # 15 X/C= 0.768 PORT,PRESS* CP= -0.15	PORT # 16 X/C = 6.784 PORT PPESS* CP= -1.465	PORT # 17 X/C= 0.8 PORT PRESS= CP= -1.415	PORT # 18 X/C= 0.816 PORT PRESS= CP= -1.29
68 'TEH (F) 68 'TEHP (F) 617MOS PPESS (PSF) 2115,7848 TUNNEL FFESS (PSF) 2074,8728 TUNNEL Q 'PSF) 40,912 TUNNEL VELOC (FPS) 187,1977405	1997.54912	1999, 59472	2001.64032	2002.66312	2003.68592	2983.68592	2003.68592	2005.73152	2001.64032
6 M2 68 TEMP (F 67MOS PPESS (TUMMEL PFESS TUMMEL O PSF TUMMEL VELOC	POPT 6.1 X/C# 0.352 POPT PPESS* CP# -1.89	POPT 0 2 X/C= 0.34 PORT PPESS= CP= -1.84	PORT 8 3 X/C# 0.416 PORT PPESS# CP# -1.79	PORT 0.448 7/C= 0.448 PORT FFESS= CP= -1.765	POPT 8 5 X/C# 0.43 POPT PPESS# CP# -1.74	K C= 0.512 KC= 0.512 PGRT PPESS= CP= -1.74	POPT # 7 X/C* 6.544 POPT PRESS* CP* -1.74	PORT # 8 X/C= 0.576 PORT PPESS= CP= -1.69	PORT # 9 X/C# 0.608 PORT PRESS* CP# -1.79

S 301	FURT # 19 X C= 0.832 FURT PRESS= 2092.2604 CP= -1.3	FORT # 20 X C= 0.548 FORT PFESS= 2092.87408 (F= -1.24	POFI # 21 CC 0.864 FUFI FFESS 2092:87468 CF = 1.24	PORT # 22 N C= 6.88 PORT PRESS= 2095,2852 (F= -1.2	POPT # 23 X C= 0.896 FORT FRESS 2093.48776 CP= -1.18	PORT # 24 (C= 0.912 PORT FRESS= 2093,69252 CP= -1.16	POPT # 25 X C= 0.928 PORT PRESS= 2094,10144 CP= -1.12	PORT # 26 XZC= 0.944 POPT FPESS= 2094,306 CF=-1.1	POFT # 27 X/C= 0.96 PORT FPESS= 2094.51056 CP= -1.06
67 PER CENT BLADES	2687.1464	2085.1088	2087.55552	2668,57832	52 5= 2089.60112	5 58 5= 2090.41936	5 34 5= 2090.62848	5= 2091,2376	2091.64672
'n	FORT # 10 X.C= 0.64 PUPT PRESS= CP= -1.8	PORT # 11 X:C= 0.672 PORT PRESS= CP= -2	PORT # 12 X C= 0.72 PORT PPESS= CP= -1.76	POPT # 13 X/C= 0,736 PORT PRESS=, CP= -1,66	PORT # 14 X-C= 0.752 PORT PRESS= CP= -1.56	POPT # 15 X.C= 0.768 PORT PRESS= CP= -1.48	PORT # 16 X:C= 0.784 PORT PRESS= CP= -1.44	PORT # 17 X/C= 6.8 PORT PRESS= CP= -1.4	PORT # 18 X/C= 0.816 PORT PRESS= CP= -1.36
1 N. M.	2086, 53272	2066. 94184	2087.55552	2087,76008	2087, 96464	2087, 96464	2687, 95464	2068.57832	2687.55552
1 MZ 68 TEMF (68 TEMF (61 TUMMEL PPESS TUMMEL Q - PSI TUMMEL VELOC	POPT 0 1 > Cm 0.352 POPT PRESSM CPm -1.86	FOFT 0 2 X (= 0,384 PUFT FFESS CF= -1,82	POFT # 3 X C = 0.416 POFT PRESS CP = -1.76	POFT # 4 1./C= 0.448 FOFT PRESS= CF= -1.74	PORT # 5 27C= 0.48 PORT PRESS= CP= -1.72	FORT # 6 X/C= 0.512 FORT FRESS= CP= -1.72	POPT # 7 X/C= 0.54 PORT PPESS= CP= -1.72	PORT 8 8 X-C= 0.576 PORT PRESS= CP= -1.66	PORT # 9 X.C= 0.608 PORT PRESS= CP= -1.76

17 BLABES	FORT # 19	FURT # 20	PORT # 21 .	PORT # 22	FORT # 23	POPT # 24	FORT 0.25	POFT # 26	PORT # 27
	X-C= 0.832	X (F 0.848	X:C= 0.854	X-(= 6 88	7. (= 0.896	X-C= 0.912	N/C= 0.928	X/C= 0.944	X/Cm 0.96
	FORT PRESS= 2095.12424	PURT PRESS 2095,53336	FORT PRESS= 2035.73792	PORT PRESS= 2085,94248	PORT FRESH 2095,94248	FORT FFESS= 2090.14704	PORT PRESS= 2096.3516	PORT PPESS= 2096,55616	PORT FRESS= 2096,76072
	CP= -1.244444444	CF= -1.2	(F= -1,17777778	(F= -1,19555558	(F= -1.15555556	(F= -1.1200000000	CP= -1.11111111	CP= -1,08888889	CPm -1.06666667
67 PER CENT BLADE	FOFT # 18	POFT # 11	PORT # 12	PORT 0 13	PORT # 14	PUFI # 15	PUPT # 16	POPT • 17	PORT # 16
	X (= 0.64	X C* 6.672	N.C= 0.72	X/C= 0.736	X-C= 0.752	X/C= 6.763	X/C= 0.784	X/C= 6.8	X/Cm 0.816
	PORT PPESS= 2090.2148	PORT PRESS 2068,96744	PORT PRESS= 2090.62392	FORT PRESS= 2091.64672	PORT PRESS= 2092.2604	FORT FRESS= 2093.67864	PUFT PPESS= 2093.69232	PORT PRESS= 2093.89688	PORT PRESS= 2094.306
	CF= -1.7777778	CF=-1.91111111	CF= -1.73333333	CF= -1.62222222	CF= -1.5555556	(P= -1.46565667	CP= -1.4	CP= -1.37777778	CP= -1,33333333
2 HZ 63 TETH (F) 61 TETH (F) TUHIEL PESS (FSF) 2115,7838 TUHIEL PESS (FSF) 2106,5796 TUHIEL 0 (FSF) 9,2052 TUHIEL VELOC (FFS) 88,79568493	FUPT # 1 XXC* 6.352 POPT : 5 = 2089.88568 CF* -1.82222222	POPT # 2 X/C= 0.584 POFT PFESS= 2090.41936 CP= -1.75555556	POFT 0 3 X-C= 0.416 PORT FFESS= 2090.62392 CF= -1.73353333	POPT # 4 X-C* 6.448 POFT FFE:5* 2091.83384 CP* -1.65888889	Publ 0 5 X.C= 0.45 Publ FFESS= 2091.2376 CF= 1.656666657	POPT # 6 X C= 0.512 POPT FFESS= 2091.2376 CP= -1.656566667	POPT # : X.C* 6.544 POPT FFESS 2091.03384 CP* -1.686863889	POFT # 8 X/C # 0.576 POFT PFESS 2091.64672 CP# -1.62222222	PORT # 9 X/Cm % 602 PURT FRESs 2090,62392 CPm -1,73333333

67 PER CEUT BLADES.	PORT # 19	FORT # 20	PORT # 21	FORT # 22	POPT # 23	FORT # 24	FORT # 25	FOFT # 26	PORT # 27
	X C= 0.832	X-C= 0.848	X-C= 0.864	NAC = 0.88	X.C= 0.896	X C= 0.912	X C= 0.928	X C = 0.944	X/C= 6.96
	PORT PPESS= 2097.98606	PUFT FFESS= 2098.80632	PORT FPESS= 2099.01088	FORT FRES: 2009,21544	POPT PPESS= 2099,21544	FORT FRESS= 2099,42	FORT PRESS= 2099,62456	PORT PFESS = 2099,82912	PORT PRESS= 2099.82912
	CF= -1.289473684	CP= -1.184210526	CF= -1.157894737	CP= -1,131570947	CP= -1,131578947	CP= -1.105263158	CP= -1.078947368	CP = -1.052631579	CP= -1.052631579
67 PEK (PORT # 16	PORT # 11	PUDT # 12	FURT # 13	POPT # 14	PORT # 15	PORT # 16	PORT # 17	PORT # 18
	X.C= 0.64	X (= 0.672	X.C= 6.72	X (= 0.736	X (= 0.752	X/C = 0.768	X:(= 0.784	X C= 0.6	X/C= 0.816
	PURT PFESS= 2094,10144	FORT PRESS= 2092.87408	PURT PRESS= 2094,305	FURT PRESS= 2095,3288	PORT PRESS= 2095,94248	FOFT PRESS = 2096.76672	FORT PRESS= 2096,96528	PORT PRESS= 2097,3744	PORT PRESS= 2097.57696
	CF= -1.789473684	CF= -1.947368421	CP= -1,753157895	(F= -1.631578947	(F= -1.552631579	CP = -1.447368421	CP= :1.421652632	CF= -1,368421053	CP= -1.342105263
4 HZ 68 TEHF (F) 68 TEHF (F) TURNEL PFESS (FSF) 2105.7848 TURNEL 0 (FSF) 7.77328 TURNEL 0 (FSF) 7.77328	POPT # 1 X/C= 0.352 PORT PPE:S= 2093.48776 CP= -1.868421053	PORT # 2 X.C* 0,384 POPT FRESS 2093.89688 CF= -1.815789474	POFT # 3 X.C= 6.416 POPT FPESS= 2094,306 CP= -1.763157895	PORT # 4 X.C= 0.448 POFF FPESS= 2094,71512 CP= -1.710526316	PORT # 5 X/C= 0.43 PORT FRESS= 2094.71512 CP= -1.710526316	PGPT # 6 X C= 0.512 POPT FFESS= 2094,71512 CP= -1,710526316	POPT # ? X-(= 6.544 POPT FFESS= 2094.71512 CP= -1.710526316	POGFT # E X:C= 6.57E PORT PFESS= 2095.12424 CP= -1.657894737	POKT 0 9 X/C* 6.668 PORT PRESS* 2694.306 CP* -1.763157895

NT BLADES	POPT # 19	PORT # 20	POFT # 21	PORT # 22	POPT # 23	PORT # 24	PORT # 25	FORT # 26	PORT # 27
	X C= 0.832	X-C= 0.548	X-C= 0.864	N/C= 0.88	X C= 0.596	X/C= 0.912	X-C= 01928	X/C= 0.944	X/C= 0.96
	POPT PPESS= 2098,3972	PORT PRESS= 2093.80632	POFT FPESS= 2098.80632	PORT FRES= 2099.42	POPT PPESS= 2049.42	PORT PRESS= 2099.42	PORT FRESS= 2099.62456	PORT FFES= 2099.62456	PORT PRESS= 2099.82912
	CP= -1.236942105	CP= -1.184210536	CP= -1.184210526	CF= 1.105263158	CP= -1.105263158	CP= -1.105263158	CP= -1.078947568	CP= -1.078947368	CP= -1.052631579
67 PER CENT	PORT # 10	POKT # 11	PORT # 12	PORT # 13	FORT # 14	PORT # 15	PORT # 16	FORT # 17	PORT # 18
	X/C= 0.64	X C= 0.672	X-Cm 0.72	X-C= 6.736	X C= 0.752	X/C= 0.768	X/C= 0.784	X/C= 6.8	X/C# 0.816
	PORT PPESS= 2094,10144	POKT PRESS= 2092.87408	FORT PRESS= 2094,306	PORT PRESS= 2095.3266	PORT FPESS= 2095.94248	PORT PPESS= 2096.3516	PORT PPESS= 2096.76072	FORT FRESS= 2097,16964	PORT PRESS= 2097,57896
	CP= -1,789473684	CF= -1.947368421	CP= -1.763157895	CP= -1.631578947	CF= -1.552631579	CF= -1.5	CP= -1.447368421	CP= -1.394736842	CP* -1,342105263
6 H2 6 TEMP (F) 6THUS FEES (PSF) 2115,7848 TURNEL D (PSF) 7,77328 TURNEL U (PSF) 7,77328	POPT # 1 X*C= 0.352 POFT PRESS= 2093.69232 CP= -1.842105263	POPT # 2 X-C= 0.384 PORT PFESS= 2094.306 CP= -1.753157895	POFT # 3 X/C# 6.416 POPT PRESS 2094.51856 CP= -1.786842105	POPT # 4 X-C= 0.448 FORT PFESS= 2694.71512 CP= -1.716526316	PORT # 5 X-(= 0.48 POFT PRESS= 2095.12424 [P= -1.657894737	FUPT # 6 X C= 0.512 FORT PFESS= 2094.91968 CF= -1.684210526	POFT # 7 X/C= 0.544 POFT FFE:s= 2094,71512 CF= -1,710526316	PORT • 8 X.C* 0.576 PORT PRESS= 2095.5336 CP* -1.605263158	PORT # 9 x/C# 0.608 PORT PPESS# 2694.306 CP# -1.763157895

67 PER CENT BLADES	PORT # 19	PORT # 26	POKT # 21	PORT ● 22	PORT # 23	PORT # 24	PORT # 25	POPT # 26	PORT # 27
	X/C= 6.832	X/C= 6.548	X-(* 0.864	X*C= 6.88	X C= 0.896	%/C= 0.912	X√C= 0,928	XXC= 0.944	XXC= 0.96
	PORT PRESS= 2095.12424	PORT PRESS 2095.73792	PORT PPESS 2095.73792	PORT FRESS= 26\$5.72792	PORT PRESS= 2695.94248	PORT PRESS= 2096.14704	PORT PRESS= 2096.3516	PORT PRESS= 2896.76072	PORT FRESS= 2096.76072
	CP= -1.24444444	CP= -1.17777778	CP= -1.17777778	CF= -1.17777778	CP= -1.15555555	CP= -1.13333333	CP= -1.11111111	CP= -1.06666667	CP= -1.066666667
	PORT # 16	PORT # 11	FURT # 12	PORT # 13	PURT # 14	FORT # 15	PORT # 16	PORT # 17	PURT # 18
	x/C= 6.64	%/C= 6.672	X/C= 6.72	N/C= 0.736	x/c= 0.752	X-C= 6.768	X/C= 0.784	x/c= 6.8	X/C= 6.816
	PORT PRESS= 2090.62392	PORT PRESS= 2089.60112	PURT PRESS= 2091.03304	PORT PRESS= 2092,2664	FURT PRESS= 2093.07864	FORT PRESS= 2093,69232	PORT PRESS= 2094.10144	PORT PRESS= 2094.71512	PURT PKESS= 2094.71512
	CP= -1.7333333	CP= -1.84444444	CP= -1.68686889	CF=-1,55555556	CP= -1.46665667	CP= -1.4	CP= -1.35555556	CP= -1.26886889	CP= -1.28888589
25 HZ 68 TEMP (F) RTHOS PRESS (PSF) 2115,7848 TURHEL PRESS (PSF) 2106,5796 TURHEL Q (PSF) 9,2052 TURHEL VELOC (FPS) 88,79568493	PORT 0 1 X/C= 0.352 PORT PRESS= 2090.62392 CP= -1.73333333	POFT # 2 X-C= 0.384 POFT FFESS= 2090.82848 CP= -1.71111111	PUFT # 3 X.C= 0.416 PUFT PRESS= 2091.03304 CF= -1.688888899	PORT # 4 X/C= 6,448 POFT PRESS= 2091.2376 CF= -1.66666667	FORT # 5 X:C= 0.48 FORT FRESS= 2091.44216 CF= -1.e4444444	PORT # 6 X C = 0.512 PORT PRESS= 2091.64672 CP= -1.62222222	PORT # 7 x/c= 0.544 PORT PRESS= 2091.85128 CP= -1.6	PORT # 8 X C= 0.576 PORT FFESS= 2092.2604 CP= -1.5555556	FORT # 9 X/C= 0.608 FORT PRESS= 2091.03304 CP= -1.68888889

	2094,10144	21312.5682	2094.71512	2094.91968	2094.91968	2095.12424	2095.12424	2095.3268	2095.73792
CENT BLADES	FORT # 19 X:C= 0.832 FORT PRESS= CP= -1.12	FOPT # 20 X C= 0.848 POFT FFESS= CF= -1.06	FORT # 21 % C= 0.864 FORT PRESS CP= -1.06	: OP1 # 22 % C= 0.88 PORT PPESS= CP= -1.04	PORT # 23 X.C= 0.096 PORT PPESS= CP= -1.04	PORT # 24 X/C= 0.912 POFT PRESS= CP= -1.02	PORT # 25 X/C= 6.928 PORT PRESS= CP= -1.02	PORT # 26 X/C= 0.944 PORT PRESS= CP= -1	PORT # 27 X/C= 6.96 PORT PRESS= CP= ~6.96
67 PER CENT	2089.66112	2067.55552	2089.60112	2091,2376	2091.64672	2092, 05584	2092.46495	2093.2832	2093.69232
ın.	PORT # 19 X/C* 6.64 PORT PRESS= CP= -1.56	PORT # 11 X/C= 0.672 PORT PPESS= CF= -1.76	PORT # 12 X/C= 0.72 PORT PRESS= CP= -1.56	PORT # 13 X/C= 6.736 PORT PPESS= CP= -1.4	PORT # 14 X/C= 0.752 PORT PRESS= CP= -1.36	MOPT # 15 X/C= 6.768 PORT PRESS= CP= -1.32	PURT # 16 X C= 0.784 PORT PRESS= CP= -1.28	PORT # 17 X/C= 0.8 PORT PRESS= CP= -1.2	POFT # 18 X/C= 0.816 PORT PRESS= CP= -1.16
30 HZ 68 TEHP (F) 87MOS PRESS : PSF) 2115,7848 TUNNEL PRESS : PSF) 2105,5568 TUNNEL 0 (PSF) 10,228 TUNNEL VELOC (FPS) 93,59867025	2089.60112	2090.01024	2090.41936	2090.62392	2090,62392	2090,62392	2090.62392	2691,63384	2089.60112
30 HZ 68 TENP (67 TENP (67 TOWNEL PFESS TOWNEL 0 (PS) TOWNEL 0	PORT # 1 X-C= 0.052 PORT PRESS= CF= -1.56	POFT # 2 X/C* # 384 PORT PPESS* CP= -1.52	POPT # 3 X/C= 0.416 PORT PRESS= CP= -1.48	PORT # 4 X/C= 0.448 POPT PPESS= CP= -1.46	PORT # 5 X/C= 0.48 POFT PRESS= CP= -1.46	PORT # 6 XXC= 0.512 POPT PPESS= CP= -1.46	FORT # 7 X/C= 0.544 POPT PRESS= CP= -1.46	PORT # 8 X/C= 0.576 PORT PPESS= CP= -1.42	PORT 8 9 X/C= 0.608 PORT PRESS= CP= -1.56

PER CENT BLADES	POFT # 19	PORT # 20	PORT # 21	POPT # 22	FORT # 23	PUFT # 24	FORT # 25	PORT # 26	PORT # 27
	X/C= 0.832	X-(= 0.848	X C= 0.864	X C= 0.88	X C= 0.896	X*C= 0.912	X Cm 0.920	N C= 0.944	K/C* 0.96
	PORT PRESS= 2096.76072	PORT PPESS 2096.96528	PORT PRESS= 2097.16984	PORT FRESS= 2097.57896	POPT FPESS= 2097,78352	POST PPESS= 2097.78352	PORT PPESS 2097.98808	PORT PRESS= 2098.19264	FORT PRESS* 2098:3972
	CP= -1.066666667	CP= -1.04444444	CP= -1.02222222	CF= -0.97777778	CP= -0.95555556	CF= -0.955555555	CPm -0.93333333	CP= -0.91111111	CP* -0.88688869
67 PER	PORT #. 16	PORT # 11	FORT # 12	PORT # 13	PORT # 14	PORT # 15	PORT # 16	PORT # 17	PORT # 18
	X/C= 0.64	X-C= 0.672	X/C= 0.72	X/C= 6.736	X-C= 0.752	X/C= 0.768	X.C= 0.784	X.C= 0.8	X/C= 0.816
	PORT PRESS= 2092.66952	PORT PRESS= 2091.64672	FORT PRESS= 2092.66952	PORT PRESS= 2093.69232	PORT PPESS= 2094.71512	PORT PRESS= 2095.12424	PORT PRESS= 2095.32E8	PORT PRESS= 2095.73792	PORT PRESS= 2096.3516
	CP= -1.51111111	CP= -1.62222222	CP= -1.51111111	CF=-1.4	CP= -1.28888889	CP= -1.24444444	CP= -1.22322222	CP= -1.17777778	CP= -1.111111111
40 HZ 63 TEMP (F) 611005 FPESS (PSF) 2115,7848 TUNNEL FFESS (PSF) 2106,5796 TUNNEL U (FSF) 9,285,2 TUNNEL VELOC (FPS) 88,79568493	PUPT # 1 X-(* 0,352 PORT PRESS* 2092,66952 CF= -1,511111111	POPT # 2 X.C= 0,384 POPT PFESS= 2092,87408 CF= -1,488888899	PORT # 3 X-C= 0.416 PORT FFESS= 2093.07864 CP= -1.46666667	PORT # 4 2/C= 0.448 POPT PEESS 2093,48776 CP= -1.42222222	FOFT # 5 Xxf = 0.48 FOFT FFESS = 2093.69232 CF= -1.4	POPT # 6 X*C= 0.512 PORT PRESS= 2093.69232 CP= -1.4	POPT # 7 X/C # 6.544 FORT PFESS 2093.69232 CP= -1.4	PORT # 8 7×C= 0.57¢ POPT PPESS= 2094.306 (P= -1.33333333	PURT # 9 X/E 0.608 PODT PRESS 2093.07864 CP* -1.466666667

PORT # 19 X/C= 0.832 PORT PFESS= 2094,51056 CP= -1.08	PORT # 20 X.C= 0.848 PORT PRESS= 2094,71512 CP= -1.06	PORT # 21 X/C= 0.864 FORT FFES= 2094.91968 CP= -1.04	PUFT # 22 X C= 0.88 FURT PFESS= 2095.12424 CP= -1.02	PORT # 23 X.C= 0.896 PORT PRESS= 2095,3288 CP= -1	POPT # 24 X-C= 6.912 FORT FRESS= 2095,73792 CP= -0.96	PORT # 25 X-C= 0.928 PORT PRESS= 2096.14764 CP= -0.92	FORT # 26 N.C= 0.944 PORT PRESS= 2096.3516 CP= -0.9	PURT # 27 X.C= 0.96 PORT PRESS= 2096.55616 CP= -0.88
10 .64 ESS= 2089.192 6	11 .672 ESs= 2087.96464 72	12 -72 ESS= 2696.01624 52	13 .736 ESS≈ 2091.03304 42	14 .752 ESS= 2091.64672 36	15 .768 ESS= 2092,64952 26	16 .784 ESS= 2092.87468 24	17 .8 ESS= 2093,2832 2	f # 18 1 0.816 1 PRESS= 2093.69232 -1.16
		PURT # X/C= 60 PURT PRE		POR 2.2.0 POR 8	909 × × × 909 1909 1909	FORY CP. ST. ST. ST. ST. ST. ST. ST. ST. ST. ST	POX POX POS POS POS POS POS POS POS POS POS POS	P × P P P P P P P P P P P P P P P P P P
	6.000 6.000	0PT # 3 /E 0.416 0RT PFESS= 2689.192 P= -1.6	4 .0 0 .4 0 .0 0 .1 0 .0 0 .1	# 69.43 69.43 1.122 1.22	# 60 00 11 60 00 00 11 60 00 00 11	2.00 2.00 4.00 1.00 1.00	3.04 0.03 0.03 0.03 0.03 0.03 0.03 0.03	POPT # 9 X/C= 0.608 PORT PRESS= 2089.60112 CP= -1.56
	PORT # 19 X/C= 0.832 PORT PRESS= CP= -1.88	PORT # 10 **C= 0.64 **C= 0.652 **PORT PRESS= 2089.192 **C= 1.65 **PORT PRESS= 2089.192 **C= 0.652 **C= 0.652 **C= 0.652 **C= 0.652 **C= 0.652 **C= 0.652 **C= 0.654 **PORT PRESS= 2087.96464 **PORT PRESS= 2087.96464	2688.57832 PORT # 10 PORT # 10 X.C. 0.64 X.C. 0.63 PORT PRESS = 2089.192 PORT PRESS = 0.632 CF = -1.6 PORT PRESS = 2089.192 PORT PRESS = -1.08 2688.98744 PORT PRESS = 2087.96464 PORT PRESS = 0.634 PORT # 12 X.C. 0.848 PORT PRESS = -1.05 PORT # 12 X.C. 0.848 PORT # 21 X.C. 0.864 PORT PRESS = 2080.01024 PORT PRESS = 2080.01024 PORT PRESS = 2080.01024 PORT PRESS = -1.04	PORT # 10 PORT # 10 PORT # 10 PORT # 19 2088,57832 PORT PRESS= 2089,192 PORT PRESS= CP=-1.6 PORT PRESS= CP=-1.6 2088,98744 PORT FRESS= 2087,96464 PORT # 20 PORT PRESS= CP=-1.06 PORT # 12 PORT PRESS= CP=-1.06 PORT PRESS= CP=-1.06 PORT # 12 PORT PRESS= CP=-1.06 PORT PRESS= CP=-1.06 PORT # 12 PORT PRESS= CP=-1.06 PORT # 21 PORT # 15 PORT PRESS= CP=-1.06 PORT # 10 PORT # 15 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10 PORT # 10	PORT # 19	PORT # 10 PORT # 10 PORT # 19 PORT # 19 2088,57832 PORT PRESS= 2089,192 PORT PRESS= 0.632 PORT PRESS= 0.632 2088,98744 PORT PRESS= 2087,96464 PORT PRESS= 2087,96464 PORT PRESS= 2087,96464 PORT PRESS= 2087,96464 2089,192 PORT PRESS= 2087,96464 PORT PRESS= 2097 PORT PRESS= 2091,6467 PORT PRESS= 2091,6467 PORT PRESS= 2091,6467 PORT PRESS= 2091,6467 PORT PRESS= 2091,6467 PORT PRESS= 2091,6467 PORT PRESS= 2091,6467 PORT PRESS= 2091,6467 PORT PRESS= 2091,6467 PORT PRESS= 2091,6467 PORT PRESS= 2091,6467 PORT PRESS= 2091,6467 PORT PRESS= 2091,6467	PORT # 10 PORT # 10 PORT # 10 2088.57832 PORT PRESS= 2089.192 PORT PRESS= CF= 1.6 PORT # 11 PORT # 20 R/C= 0.672 PORT # 20 PORT # 20 R/C= 0.672 PORT # 20 PORT # 20 R/C= 0.672 PORT # 20 PORT # 20 R/C= 0.72 PORT # 12 PORT # 20 R/C= 0.72 PORT # 20 PORT # 20 R/C= 0.73 PORT # 20 PORT # 20 R/C= 0.736 PORT # 20 PORT # 20 R/C= 1.42 PORT # 20 PORT # 20 R/C= 0.736 PORT # 20 PORT # 20	PORT # 10 PORT # 10 PORT # 19 PORT # 19 X.CE 0.64 PORT PRESS= 2089.192 PORT PRESS= 2087.96464 PORT PRESS= 2088.98744 PORT # ESS= 2087.96464 PORT # ESS= 2087.96464 PORT # ESS= 2089.192 PORT # ESS= 2087.96464 PORT # ESS= 2087.96464 PORT # ESS= 2089.192 PORT # ESS= 2091.03304 PORT # ESS= 2098.61624 PORT # ESS= 2089.60112 PORT # FRESS= 2091.03304 PORT # ESS= 2091.04662 PORT # ESS= 2090.01024 PORT # FRESS= 2091.64672 PORT # ESS= 2091.64672 PORT # ESS= 2090.01024 PORT # FRESS= 2091.64672 PORT # FESS= 2091.64672 PORT # FESS= 2090.01024 PORT # FRESS= 2092.66952 PORT # FESS= 2092.66952 PORT # FESS= 2090.01024 PORT # FESS= 2092.66952 PORT # FESS= PORT # FESS= 2090.01024 PORT # FESS= 2092.66952 PORT #

67 PER CENT BLADES	FORT # 19	POFT # 20	POPT # 21	FURT # 22	FORT # 23	FORT # 24	FORT # 25	POPT # 26	POPT # 27
	X/C= 0.832	X-C= 0.848	X-(= 0.864	X C= 0.88	X (= 0.7%	% (= 0.912	X C= 0.928	X/C= 0.944	X/C= 6,96
	FORT FRES= 2098.60176	POFT PFES= 2098.80632	PORT PRESS 2098.80632	X PPESS= 2099.01088	FORT FPEss = 2099,21544	FORT PRESS = 2099.21544	16984 FORT PRESS= 2099.42	PORT PRESS= 2099.42	PORT PRESS= 2899,82912
	CP= -1.210526316	CF= -1.184210526	(P= -1.184210526	CF= -1.157894737	CF= -1.131578947	CP= -1.131578947	CP= -1.105263158	CP= -1.105263158	CP= -1,652631579
	PORT # 10	POET # 11	PORT # 12	POFT # 13	FORT # 14	PORT # 15	PORT # 16	FURT # 17	PORT # 18
	X/C= 0.64	X C= 0.672	X.C= 0.72	XV(= 0.736	X C= 0.752	X C= 0.768	X C= 6.784	X-C= 0.8	X/C= 6.816
	PORT FPES= 2094.5	POFT FFES= 2093.283	PORT PPESS= 2094,71512	PORT PRESS = 2095.73792	PORT PRESS= 2090.55616	PURT FRESS= 2096.96528	PORT PRESS= 2097.1698:	PORT PRESS= 2097,3744	PORT PRESS= 2098.19264
	CF= -1.736942105	CF= -1.894736842	CF= -1.710526316	(P= -1.578947365	CP= -1,47368421	CP= -1.421052632	CP= -1.39473E842	CP= -1,368421053	CP= -1.263157895
60 HZ 63 TEMP (F. 61 THOS PRESS (FSF) 2115.7848 TURNEL PFESS (FSF) 2198.01152 TURNEL 0 (FSF) 7.77328 TURNEL VELOC (FPS) 81.59760333	POFT # 1 X/C* 0,552 POPT PPESS* 2094.10144 CF* -1,789473684	POPT # 2 X.C* 0.384 PORT PRESS= 2094.51056 LP=-1.736642105	FOFT # 3 X-C= 0.416 FORT FFE35= 2094.71512 CP= -1.710526316	PORT N 4 X/C= 0.448 PORT PRESS= 2094.91968 CP= -1.634210526	PORT # 5 X/C= 0.48 PORT PPE:S= 2095.12424 CP= -1.657894737	POPT # 6 X/C= 0.512 POFT FRESS= 2095.12424 CF= -1.657894737	POFT # 7 X/C= 6.544 POFT FPESS* 2095,12424 CF* -1.657694737	POPT # 8 X/C= 0.576 PORT PPESS= 2095.73792 CP= -1.578947368	PORT # 9 X-C* 0.608 PORT PPESS* 2694.71512 CP* -1.710526316

67 PER CENT BLANCS	PORT # 19	FUPT # 20	POPT # 21	FURI # 22	FORT # 22	FORT # 24	PORT # 25	PORT # 26	PORT # 27
	N.C. 6,632	X-C= 0.848	X.C= 0.864	X C= 0.88	% C= 0.894	X (* 0.912	X C= 0.928	X.C* 0.944	X-C= 0.96
	FURT PRESS 2092,05584	FORT FPESS= 2092.46496	PORT PRESS= 2092.46496	POFT FRESS = 2092, 66952	FORT FPESS= 2092,87408	FORT FRESS 2093.07864	PORT FRESS= 2093.2852	PORT PFESS 2093,2832	PORT PRESS= 2093.69232
	CP= 1	CP= -0.965517241	CP= -0.965517241	CF= -0,948275862	CF= -0.981034483	CP= ~0.913793103	CF= -0.896551724	CP* -0,896551724	CP= -0.662066956
	PORT # 10	PORT # 11	PORT # 12	PORT # 13	PORT # 14	PORT # 15	PORT # 16	PORT # 17	PORT # 18
	X/C= 0.64	X/Cm 6.672	X/Cm 6.72	X/C= 0,735	X/C= 0,752	X/C= 0.768	X/C= 6.784	X/C= 0.6	X/Cm 0.816
	PORT PRESS= 2087.96464	PORT FRESS= 2086.32816	PORT FRESS= 2087,55552	PORT PRESS= 2688,98744	PORT PRESS= 2089.60112	PORT FFESS= 2090.01024	PORT FRESS= 2090.41936	PORT FRESS= 2091.03304	PORT PRESS= 2091.44216
	CP= -1.344827586	CP= -1.48275862	CPm -1.379310345	CP= -1,258620690	CF= -1,206896552	CP= -1.172413793	CP= -1.137931034	CP= -1.086206897	CP= -1.051724138
80 MZ 68 TENP (F) ATMOS PFESS (PSF) 2115,7848 TUNNEL PFESS (FSF) 2183,92832 TUNNEL 0 (PSF) 11.86446 TUNNEL VELOC (FFS) 100.8098684	POPT # 1 X.C. 6.352 POFT PFESS 2088.37376 CP* -1.310344828	PORT # 2 X/C* 0.364 POFT FRESS 2088.78288 (P* -1.275862069	POPT # 3 X/C= 0.416 PORT PRESS= 2089.192 CP= -1.24137931	POPT # 4 X.C= 0.448 POFT FFES= 2089.60112 CF= -1.206896552	PORT # 5 X-C= 0.48 POFT FFESS= 2089.60112 CF= -1.206896552	FORT 0 6 X/C= 0.512 PORT PRESS= 2089.60112 CP= -1.206896552	PORT # 7 X/C* # 6.544 PORT PFESS = 2689.60112 CF= -1.206896552	PORT # 8 X/C= 0,576 FORT PRESS= 2090.01024 C&= -1.172413793	PORT 8 9 X/C= 0.606 PORT PRESS= 2088.37376 CP= -1.310344626

r BLARES	PURT # 19	POPT # 20	Prikt # 21	POP1 # 22	POPT # 25	POFT # 24	PORT # 25	PURT # 26	PORT # 27
	% C= 0.832	X C= (1.848	X (= 0.864	X-0x 0.88	XY = 0.896	XYC 0.912	X C= 4.928	X-C= 6.944	X-C= 0.96
	FURT FPESS= 2096.76672	FOPT PFES= 2097.16984	POFT PRESS 2097.16964	F(-1 PPESS 2097,57896	POPT PFESS 2097,57896	PORT PRESS 2097,57896	PORT PFESS 2097.78352	PORT PRESS= 2697.98808	PORT PRESS= 2098.3972
	CP= -1.000005007	CP= -1.020122222	CF* -1.02022222	CP= -0.97777778	CP= -0.9777778	CP= -0.97777778	CP= -0.95555556	CP= -6.93333333	CP= -0.888888889
67 PER CENT BLADE	PORT # 10	PORT # 11	PORT # 12	PORT # 13	PORT # 14	PORT # 15	POPT # 16	POPT # 17	PORT # 18
	X.C= 0.64	X.C= 0.672	X C* 0.72	X×C= 0.736	% (* 0.752	X.C= 6.768	X/C= 6.784	X/C= 0.8	X/C= 0.816
	PORT FRESS= 2092.66952	PORT PPESS= 2091.44216	PORT PPESS 2093.07864	PORT PRESS= 2094.10144	POFT PRESS 2094.71512	FORT PRESS= 2095.12424	PORT FPESS= 2095,5336	PORT FRESS= 2095,94248	PORT PRESS= 2096.3516
	CP= -1.51111111	CF= -1.64444444	(P= -1.46666667	(P= -1.35555556	CF* 1.28886889	CP= -1.24444444	CP= -1.2	CP= -1.15555556	CP= -1,11111111
96 HZ 68 TEMP (F) 61 THINGE PPESS (FSF) 2115,7848 THINGE Q (FSF) 9.206,5796 THINGE Q (FSF) 9.2052 THINGE VELOC (FPS) 88,79568493	POFT # 1 X/C* 0.352 POFT FFES= 2092.2604 CP* ~1.55555556	PORT # 2 %/C* 6.384 PORT PRESS= 2092.66952 CP* -1.51111111	PORT # 3 X-(* 6.416 POPT PPESS 2093.07864 CP* -1.46666667	POFT # 4 X/C= 6.448 POFT PRESS= 2693.2832 CF= -1.44444444	PORT # 5 %/C= 0.48 POPT PFESS= 2093,69232 CF= -1.4	POPT # 6 X.C= 6.512 PORT PFESS= 2093.48776 CP= -1.42222222	PORT # 7 X.C= 0.544 POPT PRES= 2093.48776 CP= -1.42222222	POPT 8 X/C= 0.576 POFT PRESS= 2093.89688 CP=-1.37777778	PORT # 9 x/c= 0.608 PORT PRESS= 2092.66952 CP= -1.51111111

IT BLADE'S	PORT # 19	FORT # 20	PORT # 21	PUPT # 22	POPT # 23	POFT # 24	POFT # 25	POFT # 26	POFT # 27
	X-C= 0.832	% C= 0.646	X C= 0.864	WXC= 0.88	% C= 0.896	X C= 0.912	X 'C= 0,928	X/C= 0.944	X/C= 0.96
	PORT PPESS= 2098.60176	FORT PPESS= 2099.01088	PORT PRESS 2099.01W88	POPT PPESS= 2094.21544	POPT PPESS= 2099.21544	PORT FFESS= 2099.42	PORT PFESS= 2099.62456	PORT PRESS= 2099.62456	PORT PRESS= 2099, 82912
	CF= -1.210526316	CP= -1.157894737	CP= -1.157894737	CP= -1.131578947	CP= +1.131578947	CP= -1.105263158	CP= -1.078947368	CP= +1.076947368	CP= -1, 052631579
67 PER CENT BLADE	PORT # 10	PORT # 11	POKT # 12	PORT # 13	PORT # 14	FURT # 15	FURT # 16	FORT # 17	FORT # 18
	X/C= 0.64	X/C= 0.672	X-C= 0.72	X-C= 6.736	N/C= 0.752	X/C= 6.768	X C= 0.784	X-C* 0.8	X-C= 0.816
	PORT PRESS= 2094.51056	PORT FRESS= 2093,69232	PORT PRESS= 2094.91968	PORT FRESS= 2095,94248	PORT PPESS= 2096,3516	PORT PRESS= 2096,96528	FURT PRESS= 2097,16984	PORT PRESS= 2097,78352	PORT PRESS= 2098,19264
	CF= -1.736842105	CP= -1,842105263	CP= -1.684210526	CF= -1.552631579	CF= -1.5	CF= -1.421052632	CF= -1,394736842	CP= -1.315789474	CP= -1,263157895
160 HZ 68 TEHF (F) ATHNS FRES (PSF) 2115,784S TURNEL PFESS (PSF) 2106,01152 TURNEL Q (PSF) 7,77328 TURNEL VELOC (FPS) 81,59760333	PORT 0 1 X/Lm 0.352 POPT PPESS= 2094.10144 CP= -1.789473684	PURT # 2 2/C= 0.384 PURT PRESS= 2094.51056 CP= -1.736842105	POPT 0 3 x/C= 0.416 POPT PPESS= 2094.71512 CF= -1.710526316	POPT 0 4 X-C= 0.448 PORT PPESS= 2694,91968 CF= -1.634210526	PORT • 5 X/C= 0.48 POFT PRESS= 2095.3288 CP= -1.631578947	POPT # 6 X/C= 0.512 POPT PPESS= 2095.12424 CP= -1.657894737	FORT 0.544 X.C= 0.544 PORT PRESS= 2095.12424 (F= -1.557294737	POPT # 8 X-C= 6.576 POPT PPESs 2095.73792 CP= -1.578947368	PORT # 9 X/C= 0.E08 PORT PRESS= 2094.71512 SP=-1.718526316

	2111.812573	2112.221693	2112.221693	2112.630813	2112.630813	2112.630813	2113.039933	2113,039933	2113,244493
98 PER CENT BLAKES	POFT # 19 X/C= 0.832 POFT PRESS= CP= ~1.35	POPT # 20 XYC= 0.848 POPT PRESS= CP= 1.25	POFT # 21 X/C= 0.864 POFT PPESS= CP= 1.25	POPT # 22 X/C= 0.88 POPT PRESS= CP= -1.15	PORT # 23 X-U= 0.896 PORT PRESS= 12 CP= -1.15	FORT # 24 X C= 0.913 PORT PPESS= CP= -1.15	PORT # 25 XXX 0.928 RORT PRESS CP= -1.05	PORT # 26 X 0 = 0,944 PORT PRESS = CF = 1,05	PORT # 27 X/Cm 0.96 PORT PPESS= CP= -1
98 PER CI	2169.971533	2169.153293	2169.971533	2110.585213	2116.789773	2111.198893	2111.403453	2111.403453	2111.812573
	PORT # 10 X.C= 0.64 PORT PRESS= CF= -1.8	PORT # 11 %/C= 0.672 PORT PRESS= CP= -2	POPT # 12 X/C= 0.72 PORT PRESS= CP= -1.8	PORT # 13 X-C= 0.736 PORT PRESS= 0.0736 CF= -1.65	PORT # 14 X-C= 0.752 PORT PFESS= CP= -1.6	PORT # 15 X C= 0,768 POPT PRESS= 2 CP= -1.5	POPT # 16 X/C# 0.784 POET FRESS CP= -1.45	PORT # 17 X/C= 0.8 PORT PRESS= CP= -1.45	PORT # 18 X/C= 0.816 PORT PRESS= CP= -1.35
HZ TEMP (F) 5 PRESS (PSF) 2121,426693 EL PRESS (PSF) 2117,335693 EL Q (PSF) 4,0912 EL VELOC (FPS) 59,34186193	2189.153293	2189.357853	2169.766973	2110.176093	2110.176093	2110.176093	2116.176093	2110.585213	2109, 971533
2 NZ 72 TEMP (1 3THOS PRESE TURNEL PPESS TURNEL Q (PS)	PORT # 1 X/C= 0.352 POPT PRESS= CP= -2	PORT # 2 X Cm 6.384 PORT PPESS# CP* -1.95	PORT # 3 X/C# 0.416 PORT PRESS# CP= -1.85	POPT # 4 X 'C # 0.448 POPT PPESS# CP* -1.75	POPT # 5 X-(= 0.48 POPT PPESS= CP= -1.75	POPT # 6 X/C= 0.512 FORT PPESS= CP= -1.75	POPT # 7 X-C# 0.544 POPT PRESS= CP# -1.75	POPT # 8 X/C= 0.576 PORT PRESS= CP= -1.65	PORT # 9 X/C* 0.608 PORT PRESS* CP* -1.8

TABLE III (continued)

T BLADES	PORT # 19	PGF1 # 20	POFT # 21	POFT # 22	PORT # 23	POFT # 24	PORT # 25	PORT # 26	POKT 0 27
	X C= 6.832	X C= 0.648	X-C= 0.864	X C= 0.88	X-C= 0.896	X C= 0.912	X (= 0.928	X/C= 0.944	X/C= 0.96
	PORT PRESS= 2119,581293	PGF1 FRES= 2119.796413	PORT PRESS= 2119.796413	PuRT FPESS= 2119.796413	PORT PRESS= 2119,994973	PORT PRESS= 2119.994973	PORT PRESS= 2119.994973	FORT PRESS= 2119.994973	PORT PRESS= 2120.199533
	CF= -1	CF -0.6	CF= -0.6	CF= -6.6	CP= -0.4	CP= -0.4	CP= -0.4	CP= -0.4	CP= -0.2
98 PER CENT BLADES	PORT # 10	PORT # 11	FÜRT # 12	POPT # 13	PORT # 14	FOPT 0 15	PORT # 16 '	PORT # 17	PORT # 18
	X-C= 0.64	%/C= 0.672	X/C= 0.72	X C= 0,736	X-C= 0.752	X/C= 0.766	X C= 0.784	X/C= 6.E	X/C= 6.816
	POPT PRESS= 2118.358493	PORT PRESS= 2118.358493	PORT PRESS= 2118.563653	PORT FRESS= 2116,767613	PORT FRESS= 2119.176733	PORT PRESS= 2119.176733	PORT FRESS= 2119.381293	PURT PPESS= 2119.361293	PORT PRESS= 2118.972173
	CP= -2	(P= -2	CP= -1.8	CP= -1,4	CF= -1.2	CP= -1.2	CP= -1	CP= -1	CP= -1.4
72 TEMP (F) ATMOS PPESS (PSF) 2121.426893 TUNNEL PRESS (PSF) 2120.404093 TUNNEL 0 (PSF) 1.0228 TUNNEL VELOC (FPS) 29.67093096	POPT 0 1 X/C= 0,352 POFT FFESS= 2119,790413 CF= -2.6	POPT 2 X/Cm 0.384 POPT PPESS= 2117.949373 CP= -2.4	POFT # 3 X/C= 0,416 PORT FRESS= 2116,153933 CF= -2,2	PORT # 4 >>C= 0.448 PORT PPESS= 2118.358493 CF= -2	POPT # 5 X'C= 6.48 POPT PPESS= 2118,358493 (P= -2	POPT # 6 X/C= 0.512 POFT FPESS= 2118.358493 EF= -2	POPT # 7 X/C= 0.544 PORT PRESS= 2116.358493 CF= -2	2118.356493	PORT 8 9 X/C= 0.608 PORT PRESS= 2118.356493 CP= -2

	2120.404093	2120.404693	2128.464093	2120.608653	2120,668653	2120.608653	2120.608653	2120.813213	2120.813213
BLANES	FORT # 19 X C= 0.832 PURT PRESS= CF= 0	PUFT # 20 %:C= 0.848 PORT PPESS= CP= 0	FOP1 # 21 2/C # 0.554 FOF1 PPESS= CP= 0	POPT # 22 XVC= 0.88 POPT PRESS= CP= 0.2	FORT # 23 X/C 0.896 PORT PPESS= CP= 0.2	PORT # 24 X/C= 0.912 PORT PRESS= CP= 0.2	FOPT # 25 X/C= 0.928 PORT PPESS= CP= 0.2	PORT # 26 X/C# 0.944 PURT PRESS CP# 0.4	PORT # 27 X/C= 0.96 PORT PRESS= CP* 0.4
98 PER CENT BLADES	2119.381293	2119.381292	2119,361293	2119.796413	2119.798413	2119.796413	2119.790413	2119,790413	2119.381293
	PORT # 10 X Cm # 0.64 PURT PRESS= CP= -1	PORT # 11 X/C= 0.672 PURT PRESS= CP= -1	PORT # 12 X/C= 0.72 PORT PRESS= CP= -1	PORT # 13 X/C= 0.736 PORT PRESS= CP= -0.6	PORT # 14 X/C= 0.752 PORT PRESS= 2 CP= -0.6	POPT # 15 X/C= 0.768 PORT PRESS= CP= -0.6	PORT # 16 X/C= 6.784 PORT PRESS= CP= -0.6	PORT # 17 X/C= 0.8 PORT PRESS= CP= -0.6	PORT # 18 X/C= 0.816 PORT PRESS= CP= -1
6 M2 72 TEMP (F) MTHOS PRESS (FSF, 2121.426893 TUNHEL PPESS (FSF, 2120.404093 TUNHEL O (PSF) 1.6228 TUNHEL O (FSF) 2.628	1 .352 Ess= 2118.972173 4	# 2 6.384 PFESS= 2119.176733 -1.2	# 3 : 0.416 : PRESS= 2119.381293 -1	4 5.448 :ESS= 2119.381293	# 5 0.48 PPESS= 2119,381293 -1	6 8.512 PESS= 2119.381293	i 0.544 i 0.544 f FRESS* 2119.381293 -1	0 8 0.576 PRESS= 2119.381293 -1	# 9 6.608 PRESS= 2119.381293 -1
6 H 72 T 6TM05 P TURREL TURREL TURREL	PUPT # 1 X/C= 0.352 FOPT PPESS= CF= -1.4	POPT # 0. POPT # 0. POPT PPE CP = -1.2	X.C. 0 PORT PR CPs -1	POPT # 4 X×C= 0.448 PORT PRESS= CP= -1	PORT # X/C= PORT PR CP= -1	PORT # 6 X/C= 0.512 PORT PRESS= CP= -1	X.C. PORT PORT CP: -1	POFT	PORT # X/C# PORT P

TABLE III (continued)

	2115.516050	2115.925170	2115.925170	2115.925178	2116,129730	2116.129738	2116.129730	2116,129730	2116.334290
PER CENT BLHDES	PORT # 19 X/C= 0.832 PORT PRESS= 2 CP= -1.2	PORT # 20 X/C= 0,848 PORT PRESS= CP= -1	PORT # 21 X.C= 0.854 PORT PRESS= CP= -1	PORT # 22 % C= 0,88 PORT PRESS= CP= -1	PORT # 23 3 C* 0.896 POPT PPESS = 3 CP= -0.9	PORT # 24 3.0≈ 0.912 FORT PRESS= (P= -0.9	FORT # 25 37 C= 0,928 PORT PRESS= 3 (P= -0.9	PORT # 26 % C= 0.944 PORT PRESS= CP= -0.9	PORT # 27 X-C= 0.96 PORT PRESS= CP= -0.8
98 PER CEI	2114.962378	2113.879570	2114,697810	2114.962378	2115,186938	2115.516050	2115.516050	2115.516050	2115.516050
	PORT # 10 X/C= 0.64 PORT FRESS= CP= -1.5	PORT # 11 X.C= 0.672 PORT PRESS= CP= -2	PORT # 12 X/C= 0.72 PORT PRESS= CP= -1.6	PORT # 13 X/C= 0.736 PORT PRESS= 2 CP= -1.5	PURT # 14 X/C= 0.752 FURT FRESS= CP= -1.4	PORT # 15 X.C= 0.763 PORT PRESS= CP= -1.2	PORT # 16 X-C= 0.784 PORT PRESS= CP= -1.2	PORT # 17 X-C= 0.9 PORT PRESS= CP= -1.2	PORT # 18 X/C* 0.816 PORT PRESS= CP* -1.2
HZ FRESS (PSF) 2120.016370 PRESS (PSF) 2117.970770 0 (PSF) 2.0456 VELOC (FPS) 41.69792403	2114,902378	2114.902370	2114.902370	2114.302378	2114.902378	2114.902379	2114, 902370	2114.902370	2114.902378
25 M2 65 TEMP (F ATMOS PRESS (TURNEL PRESS TURNEL Q (PSP TURNEL Q (PSP	PORT # 1 X-C= 0.352 PORT PRESS= CP= -1.5	PORT # 2 2.C= 9.394 POPT PFES= CP= -1.5	POPT 8 3 X/C= 0.416 PORT PRESS= CP= -1.5	FORT # 4 X C= 0.448 PORT PRESS= CP= -1.5	POPT 0 5 X C= 0.48 POPT PPESS CP= -1.5	POPT # 6 X.C= 0,512 POPT PRESS= CP= -1.5	POPT # 7 X C= 0.544 PORT FFESSE CP= -1.5	PORT # 9 X (* 0.575 POPT PRESSE CP* -1.5	PORT # 9 X/C= 9.608 PORT PRESS= CP= -1.5

TABLE III (continued)

38 PER CENT BLHDES	PORT # 19 X-C= 0.832 PORT PPESS= 2113.265890 CP= -0.65	PORT # 20 X.C= 0.948 PORT PRESS= 2113.470450 CP= -0.5	PORT # 21 X-C= 0.864 PURT PRESS= 2113.476450 CP= -0.6	PORT # 22 3. = 0.88 PORT PFESS= 2113.478458 CP= -0.6	PORT # 23 X-C= 0.896 PORT PPESS= 2113.470450 CP= -0.6	POFT # 24 %-C= 0.912 PORT PRESS= 2113.478458 CP= -0.6	PORT # 25 X C= 0.923 PORT PRESS= 2113.470450 CP= -0.6	PORT # 26 X C= 0.944 PORT PRESS= 2113.470450 CP= -0.6	PORT # 27 X/C= 0.96 PORT PRESS= 2113.470450 CP= -0.6
38 PI	10 64 SS= 2112.856770 5	11 672 55= 2112,856778 5	12 72 55= 2112,856778 5	736 55= 2113.061330	14 752 55= 2113,061330	15 768 85= 2113.061330	16 784 55= 2113,265890 5	17 .8 ESS= 2113.265890 65	18 816 5S= 2113,265890 5
978 8178 9783	PORT # 19 X/C* 0.64 PORT PRESS* CP* -0.75	PGRT # 11 X/C= 0.672 PORT PRESS= CP= -0.75	PORT # 12 X/C* 0.72 PORT PRESS* CP* -0.75	PORT # 13 X-C= 0.736 PORT PRESS= 0.7	PORT # 14 X.C= 0.752 PORT PRESS= CP= -0.7	PORT # 15 % C= 0.768 PORT PRESS= CP= -0.7	PURT # 16 X.C= 0.784 PORT PRESS= . CP= -0.65	PORT # 17 X.C= 0.8 PORT PRESS= CP= -0.65	PORT # 18 X/C= 0.816 PORT PRESS= CP= -0.65
30 HZ 65 TENP (F) ATMUS PPESS (PSF) 2120,016370 TUNNEL 0 (PSF) 4,0912 TUNNEL 0 (PSF) 4,0912 TUNNEL VELOC (FPS) 58,96976976	2112.856770	2112.856778	2112.856778	2112.856770	2112.856770	2112.356770	2112.356770	2112.856770	2112.356770
30 HZ 65 TETIP (F 67 TETIP (F TURNEL PRESS (TURNEL 0 (PSF TURNEL 0 (PSF	PORT # 1 X/C= 0.352 POPT FFESS= CP= -0.75	PORT # 2 X.C= 8.384 PORT FRESS= CP= -8.75	PORT # 3 X/C= 0.416 PORT FPESS= CP= -0.75	POPT # 4 X/C= 0.448 PORT PPESS CP= -6.75	PORT # 5 X/C= 0.48 / POPT PRESS= CP= -0.75	PURT # 6 X/C= 0.512 PORT PPESS= 2 CP= -0.75	PORT # 7 X-C= 0.544 PORT PPESS= CP= -0.75	POPT # 8 X/C# 0.576 POPT PFESS# CP# -0.7	PORT 9 9 X/C= 0.608 PORT PRESS= CP= -0.75

TABLE III (continued)

	2 = 2113,47645d	8 = 2113,470450	4 2113.470458	= 2113,470450	6 = 2113,478450	2103.675010	8 = 2113.675010	2113.675010	. 2113,675010
CENT BLHDES	PORT # 19 X/C= 0.832 PORT FRESS= CP= -0.6	PORT # 20 X:C= 0.948 PORT FRESS= 2 CP= -0.6	PORT # 21 X/C= 3.864 PORT PPESS= CP= -0.6	FORT # 22 X-C= 0.88 PORT PRESS= CP= -0.6	PORT # 23 X.C= 0.896 PORT PFESS= 2 CP= -0.6	PORT # 24 X C= 0.912 PORT PRESS= (P= -0.55	PORT # 25 3.C= 0.928 PORT PRESS= CP= -0.55	PORT # 26 X C= 0.944 PORT PRESS= CP= -0.55	PORT # 22 X.C= 0.96 PORT PRESS= CP= -0.55
98 PER CENT	99	<u>o</u>	©	ng.			s		
8	2113,470450	2113.861338	2113.265898	2113.478458	2113.470456	2113,470450	2113,67501	2113.675818	2113,478458
	PORT # 10 X/C= 0.64 PORT PRESS= CP= -0.6	PORT # 11 %/C= 0.672 PORT PRESS= CP= -0.7	PORT # 12 %/C= 0.72 PORT PRESS= CP= -0.65	PORT # 13 X-C= 0.735 PORT PRESS= CP= -0.6	PORT # 14 X/C= 0.752 PORT PRESS= CP= -0.6	PORT # 15 % C= 0.769 PORT PRESS= 0.00	PORT # 16 %/C= 0.784 PORT PRESS= CP= -0.55	PORT # 17 2/C= 0.8 POPT PPESS= CP= -0.55	PORT # 18 X/C= 0.816 PORT PRESS= CP= -0.6
40 HZ 63 TEMP (F) ATHOS PRESS (PSF) 2120.016370 TUHNEL PRESS (PSF) 2115.925170 TUHNEL 0 (PSF) 4.0912 TUHNEL VELOC (FPS) 58.96976976	2113.470450	2113,470450	2113,476450	2113,478458	2113.478458	2113.476450	2113.470450	2113.476459	2113.478458
40 HZ 65 TEMP (ATMOS PRESS TURNEL PRESS TURNEL Q (PS)	PORT # 1 X/C= 0.352 POPT PPESS= CP= -0.6	POPT 8 2 X/C= 0.384 POPT PRESS= CP= -0.6	POPT # 3 2/C= 9.41c FUPT PPESS= CP= -0.6	PORT # 4 X/CH 0.448 PORT PFESSH OFH -0.5	FORT # 5 2.00 # 9.49 PORT PRESS OF = -0.6	FOFT # 6 //C= 0.512 POFT PRESS= 2 CP= -0.6	POPT 0 544 N.C= 0.544 POPT PPESS= CP= -0.6	POPT # 3 20 C= 0.576 POPT PPE(3*	PGFT # 9 X/C# 6.608 PORT PPESS# CP# -0.6

98 PER CENT BLIJES	PORT # 19 X/C= 0.932 15730 PORT FRESS= 2112.243090 CP= -0.727272727		PORT # 21 X-C= 0.864 15730 PURT PRESS= 2112.447650 CP= -0.631318132	PURT # 22 3.45 6.88 PURT PRESS 2112,447650 CP= -0.681818182	PURT # 23 X C= 0.896 24850 PURT PPESS= 2112.652210 CP= -0.6383636	PORT # 24 X C= 0.912 83970 PORT PFESS= 2112.856770 CP= -0.590303091		PURT # 26 X C= 0.944 88530 PORT PRESS= 2113.265890 CP= -0.5	
	PORT # 10 X/Cm 0.64 PORT PRESS= 2111.015730 CPm -1	PURT # 11 X/C= 0.572 PURT FRESS= 2110.402050 CP= -1.13636363	PORT # 12 X-C= 0.72 PORT PRESS= 2111.015730 CP= -1	FURT # 13 X-C= 0.736 PURT PRESS= 2111.424850 CP= -0.909090909	PORT # 14 %/C= 0.752 PORT PRESS= 2111.424850 CP= -0.90909909	PÚRT # 15 2.C= 0.763 PORT PPESS≈ 2111.833970 CP= -0.818181818	PORT # 16 X C= 0.784 FORT PRESS= 2111.833978 CP= -0.818191818	PORT # 17 X-C= 0.8 PORT PRESS= 2112.038530 CP= -0.772727273	PORT # 18 X-C= 0.816 PORT PRESS= 2112.038530 CP= -0.772727273
50 HZ 65 TEHP (F) 610 MINUS PRESS (PSF) 2120,016370 TURNEL PRESS (PSF) 2115,516050 TURNEL O (PSF) 4,50032 TURNEL VELOC (FPS) 61.84801630	POPT # 1 7.6 = 0.352 PORT PPESS = 2110.402050 CP= -1.136363636	PORT # 2 X-C= 0.334 POPT PPESS= 2110.402050 CP=-1.136363636	POPT # 3 X/C= 0.416 PORT PRESS= 2110.606610 CP= -1,09090309	PORT 0 4 X/C= 0.443 PORT PFESS= 2110.606618 CF= -1.09090909	FGFF # 5 X-C= 0.48 PGFFFE5s= 2110.606610 CP= -1.09040909	POPT # 6 % C= 0.512 POPT PFESS= 2110.606619 CP= -1.8989899	FORT # 7 X C= 0.544 FORT FFE:S= 2110.606610 JP= -1.09090909	FORT B 3 > C= 0.576 PORT PPESS= 2119.606610 CP=-1.09090903	PGFT # 9 x/c= 0.608 PGRT PPESS= 2110.606610 CP= -1.09090309

	2116.538850	2116.538850	2116.538850	2116,743410	2116.743410	2116.743410	2116.743410	2116,947970	2116.947978
вгарез	PORT # 19 X C= 0.832 PURT PRESS= CP= -0.7	PORT # 20 X.C# 0.043 PORT PRESS# CP= -0.7	PURT # 21 X C# 0.864 PURT PRESS# 2 CP# -0.7	FORT # 22 2/C= 0.08 PORT PRESS= CP= -0.6	PORT # 23 X f = 0.098 FORT PPESSE (P= -0.0	1967 1970 1970 1970 1970 1970 1970 1970 197	PORT # 257 20.0 0.928 PORT PRESS CP= -0.6	FORT # 15 0.044 PORT PRESS CP# -0.5	PORT # 27 X CE 6.96 PORT PRESSE CP = -6.5
98 PER CENT BLADES	2115.925178	2115,925178	2116.129730	2116,129738	2116,129738	2116.334290	2116.334290	2116.334290	2116.538850
	PORT # 10 X/C= 0.64 PORT PRESS= CP= -1	PURT # 11 X/C= 0.672 PORT FRESS= CP= -1	PORT # 12 X/C= 0.72 PORT PRESS= CP= -0.9	PORT # 13 X C= 0.736 PORT FRESS= CF= -0.9	PORT # 14 X C* 0.752 PORT PRESS* CP* -0.9	PORT # 15 % C= 0.768 PORT PPESS= 3 CP= -0.8	PORT # 16 X-C* 0.784 PORT PPESS* CP* -0.8	POPT # 12 X/C= 0.8 POPT PRESS= CP= -0.8.	PORT # 18 X/C# 0.816 PORT PRESS# CP# -0.7
60 HZ 65 TENP -F./ MTHUS PPESS PSF - 2120.016370 TUNNEL PPESS (PSF) 2117.970770 TUNNEL 0 -PSF) 2.0456 TUNNEL VELOC (FPS) 41.69792408	2115.925178	2115.925170	2115,925170	2115, 925170	2115.925178	2115,925170	2115,925179	2115.925170	2115,925178
60 HZ 65 TEMP (F 67 TEMP (F FORMEL PRESS TURNEL O (PSF TURNEL VELOC	PORT # 1 200 # 0.352 PORT PPESS# CP# -1	POPT # 2 X-C* 0.384 POPT FRESS* CP* -1	PUPI # 3 XXC= 0.416 PUPI PRESS= CP= -1	PORT # 4 X C= 0.448 PORT PPESS= (P= -1	POPT # 5 X (= 0.48 POPT PFESS= (P= -1	POPT # 60.51.2 POPT # PESSH	POFT # 7 X C* 0.544 PORT PPESS* CF* -1	FUP1 # 3 2.0* 9.576 POP1 PPESS* CP* -1	POPT # 9 X/C= 0.608 PORT PPESS= CP= -1

TABLE III (continued)

	2117, 35,098	2117, 357838	2117, 357898	2117.35,090	2117,357898	2117,561650	2117,561650	2117,788210	2117.766218
લ્લાયલ :	PORT # 19 27-C= 0.832 PURT PRESS= CP= -0.625	PORT . 20 N C* 0.848 PORT PPESS* CP* -0.625	POPT # 21 X C# 0.864 POPT PPENSE CP# -0.625	FORT # 22 X C* 0.03 FORT FRESS CP* 0.605	POPT # 13 X C= 9,895 PORT PRESS CP= -9,625	PORT # 24 :: (* 0.912 PORT PRESS CP= -0.5	PORT # 155	FORT # 100 C C C C C C C C C C C C C C C C C C	FURT # 27 % C= 0.96 PORT PRESS= CP= -0.375
98 PER CENT BLMGE:	2117.357898	2117.357098	2117,357090	2117,357098	2117,357090	2117, 357090	2117, 357090	2117.357898	2117.357090
	PORT # 10 X/C* 0.64 PURT PRESS* CP* -0.625	PORT # 11 X.C= 0.672 PORT PRESS= CP= -0.625	PORT # 12 X.C= 0.72 PORT PPESS= CP= -0.625	PORT # 13 % C= 0.736 PORT PRESS= CP= -0.625	PORT # 14 2. C= 0.752 FORT PFESS= CP= -0.025	POPT # 15 X C= 0.768 PORT PPESS= CP= -0.625	PORT # 16 X C= 0.794 POPT PPESS= (P= -0.625	PUPT # 17 X C= 0.3 PORT PRESS= CP= -0.625	PORT # 18 X-C= #.816 PORT PPESS= CP= -8.625
70 MZ 65 TEHP (F) ATINGS PPESS (PSF) 2120-016370 TUNNEL PFESS (PSF) 2110.379890 TUNNEL (PSF) 1.63648 TUNNEL (PEDC (FPS) 37.2957571	2117.357898	2117, 357898	2117.357898	2117,357890	2117.357098	2117,357899	2117,357098	2117,357090	2117, 35:090
78 MZ 65 TENP (F) TUNNEL PFESS (F) TUNNEL O (PSF)	PORT 8 1 X×C* 0, 352 PORT PRESS* CP* -0,625	PORT # 2 X C* 0, 534 PORT PRESS* CF* -0,625	PORT # 3 X-C* 0.416 POPT PRESS CP* -0.625	POPT # 4 X (* 0,443 POFT PEESS# CP= -0,625	POFF 8 5 3. Ca 0.48 POFF PEESS CP= -0.625	POPT # 6 X-C* 0,512 PORT PRESS* CP* -0,525	PORT 6 2 2.04 0.544 PORT PRESS CP= -0.625	FORT # 8 X C# 0.576 POPT PPESS# CP# -0.625	PORT # 9 X/C* 0.603 PORT PPESS* CP*-0.625

TABLE IV

PRESSURE SURVEY DATA

PRIMARY MODEL

67% Shutter Blades Installed

Ambient Conditions: Temperature = 68°F, Pressure = 30.00 inches Hg

f = 0 Hz $q = 20 \text{ cm H}_20$ f = 1 Hz q = 5.0 cm H₂0

	~				
Static Port Number	Pressure (cm H ₂ 0)	Static Port Number	Pressure (cm H ₂ 0)		
1 2 3 4	37.8	1	9.3		
2	36.8	2	9.1		
3	35.8	2 3 4	8.8		
	35.3	4	8.7		
2	34.8	5	8.6		
5 6 7 8	34.8 34.8	0	8.6		
(9	34.8 33.8	? 8	8.6		
9	33.8 35.8	9	8.3 8.8		
10	36.8	10	9.0		
11	39.8	11	10.0		
12	35.8	12	8.8		
13	34.3	13	8.3		
14	32.3	14	7.8		
15	29.3	15	7.4		
16	28.3	16	7.2		
17	25.8	17	7.0		
18	24.8	18	6.8		
19	23.8	19	6.5		
20	23.3	20	6.2		
21	22.8	21	6.2		
22	22.3	22	6.0		
23	21.3	23	5.9		
24	20.8	24	5.8		
25 26	20.3	25	5.6		
26 27	20.0	26	5.5		
27	19.8	27	5.4		

PRESSURE SURVEY DATA

PRIMARY MODEL

67% Shutter Blades Installed

Ambient Conditions: Temperature = 68°F, Pressure = 30.00 inches Hg

f = 2 Hz $q = 4.5 \text{ cm H}_20$ f = 4 Hz $q = 3.8 \text{ cm H}_20$

Static Port	Pressure	Chadda Bamb	_
Number	(cm H ₂ 0)	Static Port Number	Pressure (cm H ₂ 0)
1	8.2	1	7.1
2	7.9	2	6.9
3	7.8	3	6.7
4	7.6	4	6.5
1 2 3 4 5 6 7 8	7.5	2 3 4 5 6	6.5
6	7.5		6.5
7	7.6	7	6.5
8	7.3	? 8	6.3
9	7.8	9	6.7
10	8.0	10	6.8
11	8.6	11	7.4
12	7.8	12	6.7
13	7.3	13	6.2
14	7.0	14	5.9
15	6.6	15	5.5
16	6.3	16	5.4
17	6.2	17	5.2
18	6.0	18	5.1
19	5.6	19	4.9
20	5.4	20	4.5
21	5.3	21	4.4
22	5.2	22	4.3
23	5.2	23	4.3
24	5.1	24	4.2
25	5.0	25	4.1
26	4.9	26	4.0
27	4.8	27	4.0

PRESSURE SURVEY DATA

PRIMARY MODEL

67% Shutter Blades Installed

Ambient Conditions: Temperature = 68°F, Pressure = 30.00 inches Hg

f = 6 Hz $q = 3.8 \text{ cm H}_20$ f = 25 Hz $q = 4.5 \text{ cm H}_20$

4 - 2:0 cm 1120		4 - 4.7 cm 1.20			
Static Port Number	Pressure (cm H ₂ 0)	Static Port Number	Pressure (cm H ₂ 0)		
1	7.0	1	7.8		
2	6.7	2	7.7		
3	6.6	3	7.6		
1 2 3 4 5 6 7 8	6.5	2 3 4	7.5		
5	6.3	5	7.4		
6	6.4	6	7.3		
7	6.5	5 6 7 8	7.2		
8	6.1	8	7.0		
9	6.7	9	7.6		
10	6.8	10	7.8		
11	7.4	11	8.3		
12	6.7	12	7.6		
13	6.3	13	7.0		
14	5.9	14	6.6		
15	5.7	15	6.3		
16	5.5	16	6.1		
17	5.2	17	5.8		
18	5.1	18	5.8		
19	4.7	19	5.6		
20	4.5	20	5.3		
21	4.5	21	5.3		
22	4.2	22	5.3		
23	4.2	23	5.2		
24	4.2	24	5.1		
25	4.1	25	5.0		
26	4.1	26	4.8		
27	4.0	27	4.8		

PRESSURE SURVEY DATA

PRIMARY MODEL

67% Shutter Blades Installed

Ambient Conditions: Temperature = 68°F, Pressure = 30.00 inches Hg

f = 30 Hz $q = 5.0 \text{ cm H}_20$

f = 40 Hz q = 4.5 cm H₂0

~				
Static Port Number	Pressure (cm H ₂ 0)	Static Port Number	Pressure (cm H ₂ 0)	
1 2 3 4 5 6 7 8 9	7.8	1	6.8	
2	7.6	2	6.7	
ز	7.4	3	6.6	
4	7.3	1 2 3 4 5 6 7 8 9	6.4	
2	7.3	5	6.3	
0	7.3	6	6.3	
′	7.3	7	6.3	
0	7.1	8	6.0	
40	7.8	9	6.6	
11	7.8		6.8	
11	8.8	11	7.3	
12	7.8	12	6.8	
13	7.0	13	6.3	
14	6.8	14	5.8	
15 16	6.6	15	5.6	
10	6.4	16	5.5	
17 18	6.0	17	5.3	
10	5.8	18	5.0	
19 20	5.6	19	4.8	
21	5.3	20	4.7	
22	5.3	21	4.6	
23	5.2	22	4.4	
24	5.2	23	4.3	
25	5.1	24	4.3	
25 26	5.1	25	4.2	
27	5.0	26	4.1	
~(4.8	27	4.0	

PRESSURE SURVEY DATA

PRIMARY MODEL

67% Shutter Blades Installed

Ambient Conditions: Temperature = 68°F, Pressure = 30.00 inches Hg

f = 50 Hz q = 5.0 cm H₂0 f = 60 Hz $q = 3.8 \text{ cm H}_20$

	L	4			
Static Port Number	Pressure (cm H ₂ 0)	Static Port Number	Pressure (cm H ₂ 0)		
1 2 3 4	8.3	1	6.8		
2	8.1	2	6.6		
3	8.0	1 2 3 4 5 6 7 8 9	6.5		
4	7.8	4	6.4		
5 6 7 8	7.6	5	6.3		
6	7.6	6	6.3		
?	7.6	7	6.3		
8	7.2	8	6.0		
9	7.8	9	6.5		
10	8.0	10	6.6		
11	8.6	11	7.2		
12	7.6	12	6.5		
13	7.1	13	6.0		
14	6.8	14	5.6		
15 16	6.3	15	5.4		
16	6.2	16	5.3		
17	6.2 6.0	17	5.2		
18	5.8	18	4.8		
19	5.4	19	4.6		
20	5.3	20	4.5		
21	5.2	21	4.5		
22	5.1	22	4.4		
23	5.0	23	4.3		
24	4.8	24	4.3		
25	4.6	25	4.2		
26	4.5	26	4.2		
27	4.4	27	4.0		

PRESSURE SURVEY DATA

PRIMARY MODEL

67% Shutter Blades Installed

Ambient Conditions: Temperature = 68°F, Pressure = 30.00 inches Hg

f = 70 Hz $q = 3.8 \text{ cm H}_20$ f = 80 Hz $q = 5.8 \text{ cm H}_20$

	~		4
Static Port Number	Pressure (cm H ₂ 0)	Static Port Number	Pressure (cm H ₂ 0)
1 2 3 4	6.8	1 2 3 4 5 6 7 8 9	7.6
2	6.6	2	7.4
3	6.3	3	7.2
4	6.2	4	7.0
5 6 7 8 9	6.2 6.2 6.2	5	7.0
6	6.2	6	7.0
?	6.2	7	7.0
8	5.8	8	6.8
9	6.4	9	7.6
	6.6	10	7.8
11	7.0	11	8.6
12	6.4	12	8.0
13	6.0	13	7.3
14	5.6	14	7.0
15	5.4	15	6.8
16	5.2	16	6.6
17	5.0	17	6.3
18	4.8	18	6.1
19	4.6	19	5.8
20	4.4	20	5.6
21	4.4	21	5.6
22	4.4	22	5.5
23	4.3	23	5.4
24	4.2	24	5.3
25	4.2	25	5.2
26	4.1	26	5.2
27	4.0	27	5.0

PRESSURE SURVEY DATA

PRIMARY MODEL

67% Shutter Blades Installed

Ambient Conditions: Temperature = 68°F, Pressure = 30.00 inches hg

f = 90 Hz q = 4.5 cm H₂0 f = 100 Hz $q = 3.8 \text{ cm H}_20$

•	4	_	2
Static Port Number	Pressure (cm H ₂ 0)	Static Port Number	Pressure (cm H ₂ 0)
1	7.0	1	6.8
2	6.8	2	6.6
3	6.6	3	6.5
4	6.6 6.5	4	6.4
5	6.3	5	6.2
6	6.4	6	6.3
7	6.4 6.4	7	6.3
8	6.2	8	6.0
1 2 3 4 5 6 7 8 9	6.8 6.8	1 2 3 4 5 6 7 8 9	6.0 6.5
10	6.8	10	6.6
11	7.4	11	7.0
12	6.6	12	6.4
13	6.1	13 14	5.9
14	5.8	14	5.7
15	5.6	15	5.7 5.4
16	5.4	16	5.3
17	5.2	17	5.0
18	5.0	18	4.8
19	4.8	19	4.6
20	4.6	20	4.4
21	4.6	21	4.4
22	4.4	22	4.3
23	4.4	23	4.3
24	4.4	24	4.2
25	4.3	25	4.1
26	4.2	26	4.1
27	4.0	27	4.0

PRESSURE SURVEY DATA

PRIMARY MODEL

98% Shutter Blades Installed

Ambient Conditions: Temperature = 72°F, Pressure = 30.08 inches Hg

f = 1 Hz $q = 3.0 \text{ cm H}_20$ f = 2 Hz $q = 2.0 cm H_20$

			-
Static Port Number	Pressure (cm H ₂ 0)	Static Port Number	Pressure (cm H ₂ 0)
1	5.0	1	4.0
2	4.9	2	3.9
1 2 3 4 5 6 7 8 9	4.8	1 2 3 4 5 6 7 8 9	3.7
4	4.5	4	3.5
5	4.5	5	3.5 3.5
6	4.5	6	3.5
?	4.5	7	3.5
8	4.4	8	3.3
9	5.0	9	3.6
10	5.0	10	3.6
11	5.3	11	4.0
12	5.0	12	3.6
13	4.5	13	3.3
14	4.3	14	3.2
15	4.0	15	3.0
16	3.9	16	2.9
17	3.5	17	2.9
18	3.7	18	2.7
19	3.5	19	2.7
20	3.3	20	2.5
21	3.3	21	2.5 2.3
22	3.3	22	2.3
23	3.2	23	2.3 2.3
24	3.1	24	2.3
25 26	3.0	25	2.1
26 27	3.0	26	2.1
27	3.0	27	2.0

PRESSURE SURVEY DATA

PRIMARY MODEL

98% Shutter Blades Installed

Ambient Conditions: Temperature = 72°F, Pressure = 30.08 inches Hg

$f = 4 Hz$ $q = 0.5 cm H_2O$		$f = 6 \text{ Hz}$ $q = 0.5 \text{ cm H}_2\text{O}$		
Static Port Number	Pressure (cm H ₂ 0)	Static Port Number	Pressure (cm H ₂ 0)	
1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21	1.3 1.2 1.1 1.0 1.0 1.0 1.0 1.0 1.0 1.0 0.9 0.7 0.6 0.5 0.5 0.5 0.5	1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21	0.7 0.5 0.5 0.5 0.5 0.5 0.5 0.5 0.5 0.5 0.5	
22 23 24	0.3 0.2 0.2	22 23	-0.1 -0.1	
25 26	0.2	24 25 26	-0.1 -0.1	
26 2 7	0.2 0.1	26 27	-0.2 -0.2	

PRESSURE SURVEY DATA

PRIMARY MODEL

98% Shutter Blades Installed

f = 25 Hz

Ambient Conditions: Temperature = 65°F, Pressure = 30.06 inches Hg

f = 30 Hz

 $q = 1.0 \text{ cm H}_20$ $q = 2.0 \text{ cm } H_20$ Static Port Pressure Static Port Pressure (cm H₂0) Number (cm H₂0) Number 1.555555555555444333332222 12345678911123145161718 12345678901123456 17 18 19 20 21 22 23 24 25 26 27 1.2 1.0 1.0 1.0 0.9 19 20 21 22 23 24 25 26 0.9 1.2 0.9 0.9 27

AD-A096 386 NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4 EFFECTS OF OSCILLATION FREQUENCY AND AMPLITUDE ON SEPARATION IN-ETC(U) SEP 80 M FOX

NL

END

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4
EFFECTS OF OSCILLATION FREQUENCY AND AMPLITUDE ON SEPARATION IN-ETC(U)

NL

END

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4
EFFECTS OF OSCILLATION FREQUENCY AND AMPLITUDE ON SEPARATION IN-ETC(U)

NL

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4
EFFECTS OF OSCILLATION FREQUENCY AND AMPLITUDE ON SEPARATION IN-ETC(U)

NL

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4
EFFECTS OF OSCILLATION FREQUENCY AND AMPLITUDE ON SEPARATION IN-ETC(U)

NL

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4
EFFECTS OF OSCILLATION FREQUENCY AND AMPLITUDE ON SEPARATION IN-ETC(U)

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4
EFFECTS OF OSCILLATION FREQUENCY AND AMPLITUDE ON SEPARATION IN-ETC(U)

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

AD-A096 386

NAVAL POSTGRADUATE SCHOOL MONTEREY CA F/G 20/4

AD-A096 386

AD-A0

PRESSURE SURVEY DATA

PRIMARY MODEL

98% Shutter Blades Installed

Ambient Conditions: Temperature = 65°F, Pressure = 30.06 inches Hg

f = 40 Hz $q = 2.0 \text{ cm H}_20$

f = 50 Hz $q = 2.2 \text{ cm H}_20$

7	4	-	
Static Port Number	Pressure (cm H ₂ 0)	Static Port Number	Pressure (cm H ₂ 0)
1	1.2	1	2.5
1 2 3 4 5 6 7 8 9	1.2	2 3 4 5 6 7 8	2.5
3	1.2	3	2.4
4	1.2	4	2.4
5	1.2	5	2.4
6	1.2	6	2.4
7	1.2	7	2.4
8	1.2 1.2	8	2.4
9	1.2	9	2.4
10	1.2	9 10	2.2
11	1.4	11	2.5
12	1.3	12	2.2
13	1.2 1.2	13	2.0
14	1.2	14	2.0
1 <i>5</i> 16	1.2	15 16	1.8
16	1.1	16	1.8
17	1.1	17	1.7
18	1.2	18	1.7
19	1.2	19	1.6
20	1.2	19 20	1.5
21	1.2	21	1.5
22	1.2	22	1.5
23	1.2	23	1.4
24	1.1	24	1.3
25	1.1	25	1.2
26	1.1	26	1.1
27	1.1	27	1.0

PRESSURE SURVEY DATA

PRIMARY MODEL

98% Shutter Blades Installed

Ambient Conditions: Temperature 65°F, Pressure 30.06 inches Hg

f = 60 Hz f = 70 Hz $q = 1.0 \text{ cm H}_20$ $q = 0.8 \text{ cm H}_20$

~			-
Static Port Number	Pressure (cm H ₂ 0)	Static Port Number	Pressure (cm H ₂ 0)
1	1.0	1	0.5
2	1.0	2	0.5
3	1.0	3	0.5
4	1.0	4	0.5
2 3 4 5 6	1.0	2 3 4 5 6 7 8 9	0.5
6	1.0	6	0.5
7	1.0	7	0.5
? 8	1.0	8	0.5
9	1.0	9	0.5
10	1.0	10	0.5
11	1.0	11	0.5
12	0.9	12	0.5
13	0.9	13	0.5
14	0.9	14	0.5
15	0.8	15	0.5
16	0.8	16	0.5
17	0.8	17	0.5
18	0.7	18	0.5
19	0.7	19	0.5
20	0.7	20	0.5
21	0.7	21	0.5
22	0.6	22	0.5
23	0.6	23	0.5
24	0.6	24	0.4
25	0.6	25	0.4
26	0.5	26	0.3
27	0.5	27	0.3

LIST OF REFERENCES

- 1. Karlsson, S. K. F., <u>An Unsteady Turbulent Boundary Layer</u>, Ph. D. Thesis, Johns Hopkins University, 1958.
- 2. Nickerson, R. J., The Effect of Free Stream Oscillation on the Laminar Boundary Layer on a Flat Plate, Sc. 0. Thesis, Massachusetts Institute of Technology, 1957.
- 3. Despard, R. A., <u>Laminar Boundary Layer Separation in</u>
 <u>Oscillating Flow</u>, Ph. D. Thesis, Naval Postgraduate
 School, June 1969.
- 4. Miller, J. A., <u>Transition in Oscillating Blasius Flow</u>, Ph. D. Thesis, Illinois Institute of Technology June 1963.
- 5. AGARD Conference Proceedings No. 227, <u>Unsteady Boundary</u>
 <u>Lavers. Separated and Attached</u>, Demetri P. Telionis,
 September 1977.
- 6. Allen, T. J., <u>Pressure Distribution on an Airfoil in Oscillating Flow</u>, M. S. Thesis, Naval Postgraduate School, June 1969.
- 7. Jacobs, K. J., <u>Intensity Distribution in the Oscillating</u>
 <u>Turbulent Boundary Layer on a Flat Plate</u>, M. S. Thesis,
 Naval Postgraduate School, March 1968.
- 8. Bradshaw, P., <u>Introduction to Turbulence and Its</u>
 <u>Measurement</u>, Peragamon Press, 1975.

INITIAL DISTRIBUTION LIST

		No. Copies
1.	Defense Technical Information Center Cameron Station Alexandria, Virginia 22314	2
2.	Library, Code 0142 Naval Postgraduate School Monterey, California 93940	2
3.	Department Chairman, Code 67 Department of Aeronautics Naval Postgraduate School Monterey, California 93940	1
4.	Assoc. Prof. J. A. Miller, Code 67Mo Department of Aeronautics Naval Postgraduate School Monterey, California 93940	1
5•	LT Martin Fox, USN 13125 Wilcox Road #4B4 Largo, Florida 33540	1

